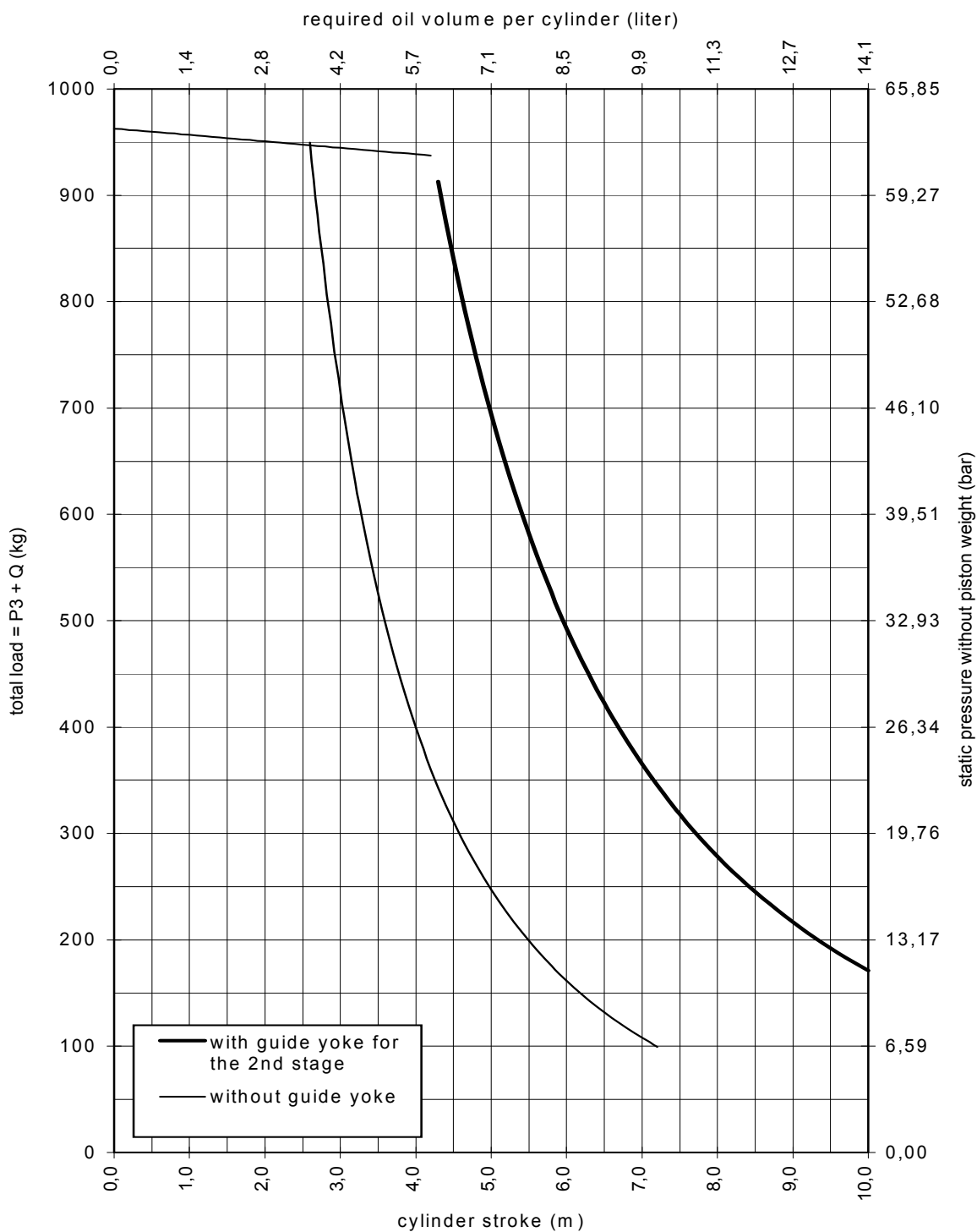




## Cylinder 3PL Grapichs

C.O.A.M.SRL RI VE/PI/ CF 00187090279 REA n. 93145  
Via Noalese Sud, 66 Pianiga 30030 VE Italia Tel +39041468966 Fax +39041468463  
e-mail: [info@coam-spa.com](mailto:info@coam-spa.com) web: [www.coam-spa.com](http://www.coam-spa.com)

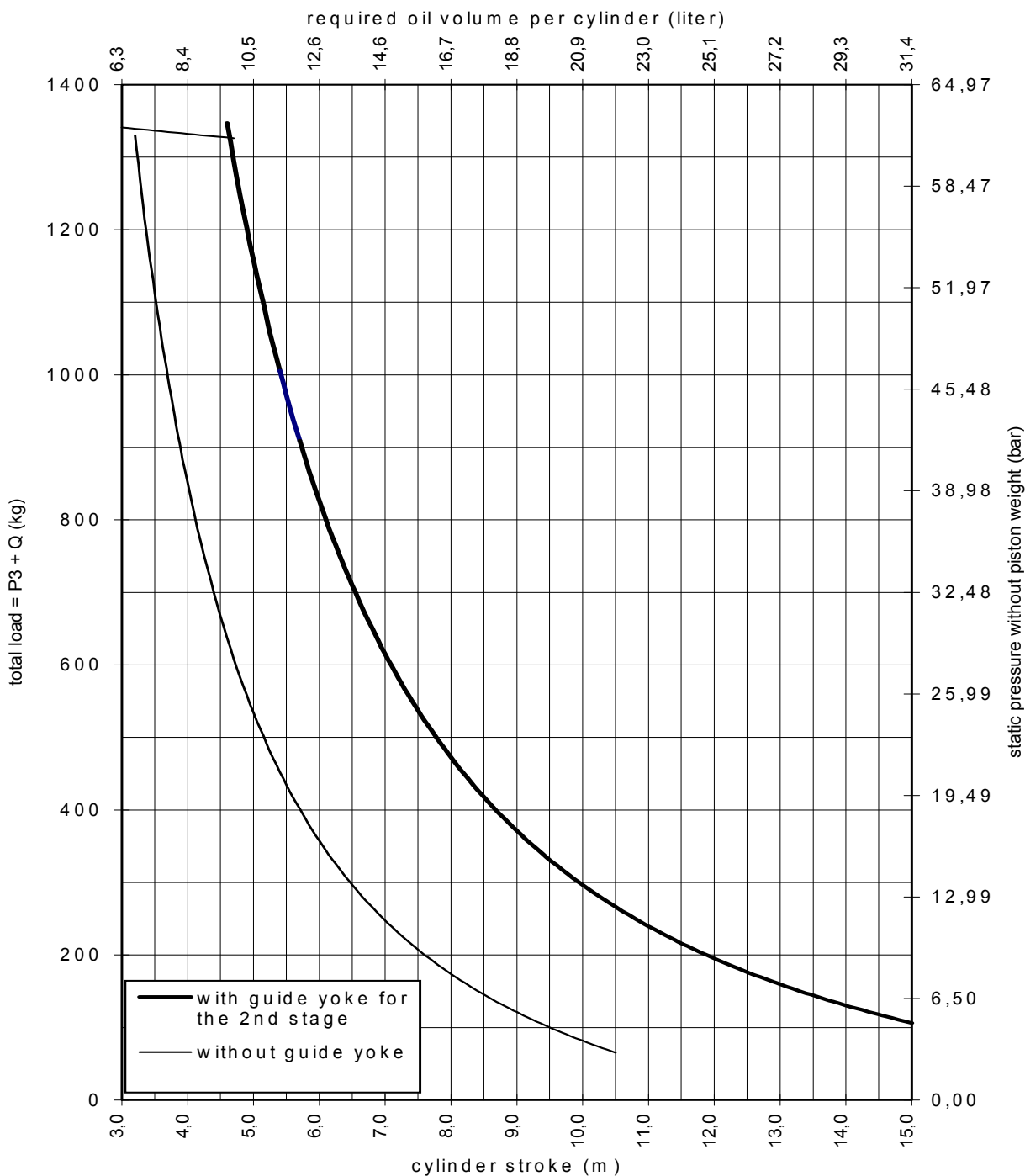
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 35/2 - VE



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 35 / 50 (mm)	Factor of excess pressure = 1,4
reference area	A = 14,897 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 11,244 (kg) (0 stroke) + 6,045 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

<b>C.O.A.M.</b> S.p.A. COMPONENTI OLEODINAMICI PER ASCENSORI E MONTACARICHI	Selection diagram for telescopic 2-stage cylinder - Central Direct System	emesso	S. A.	20/11/1998
		controllato		02/07/2002
		nullaosta		
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			di	11

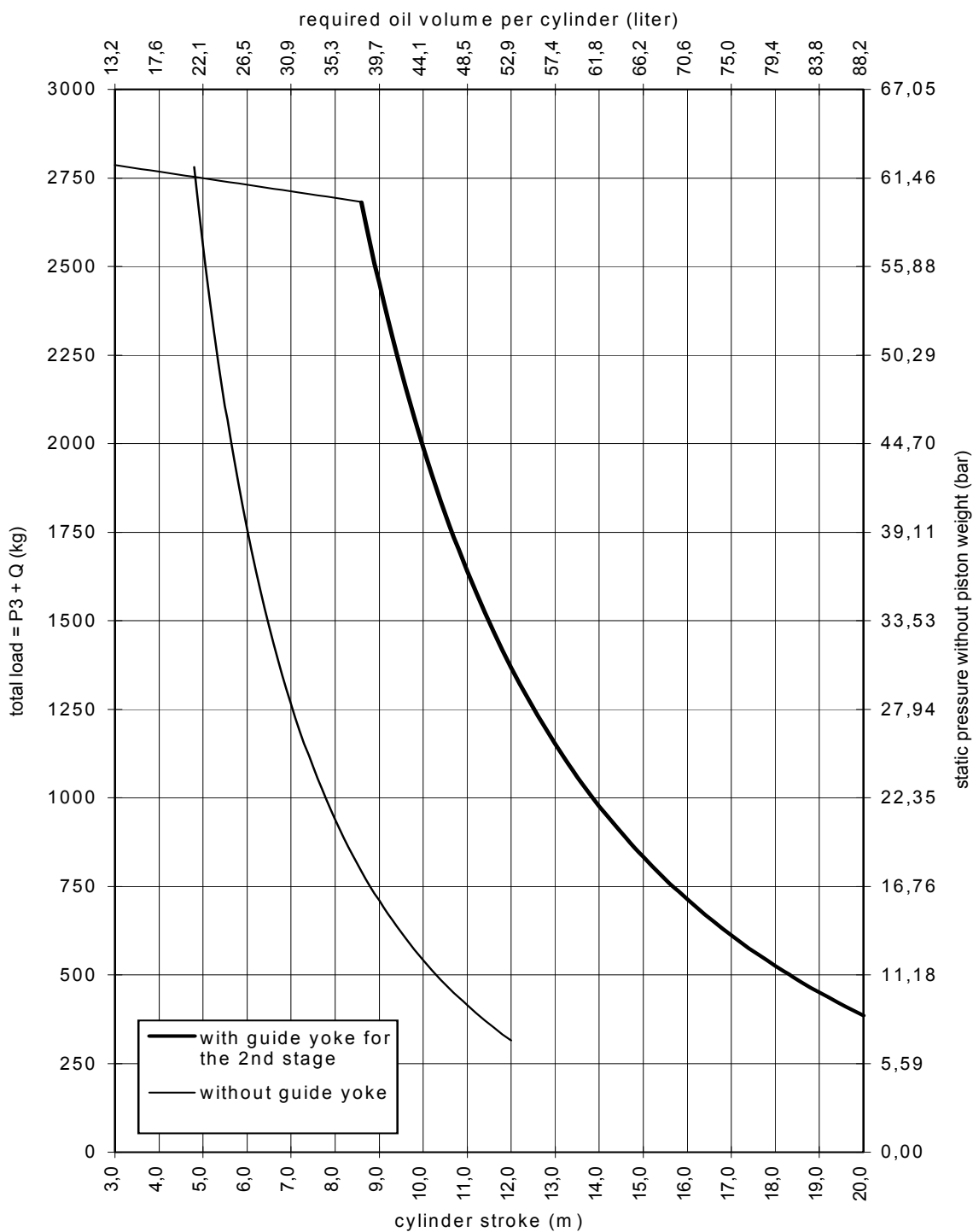
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 42/2 - VE



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 42 / 60 (mm)	Factor of excess pressure = 1,4
reference area	A = 21,138 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 14.37 (kg) (0 stroke) + 8,8 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

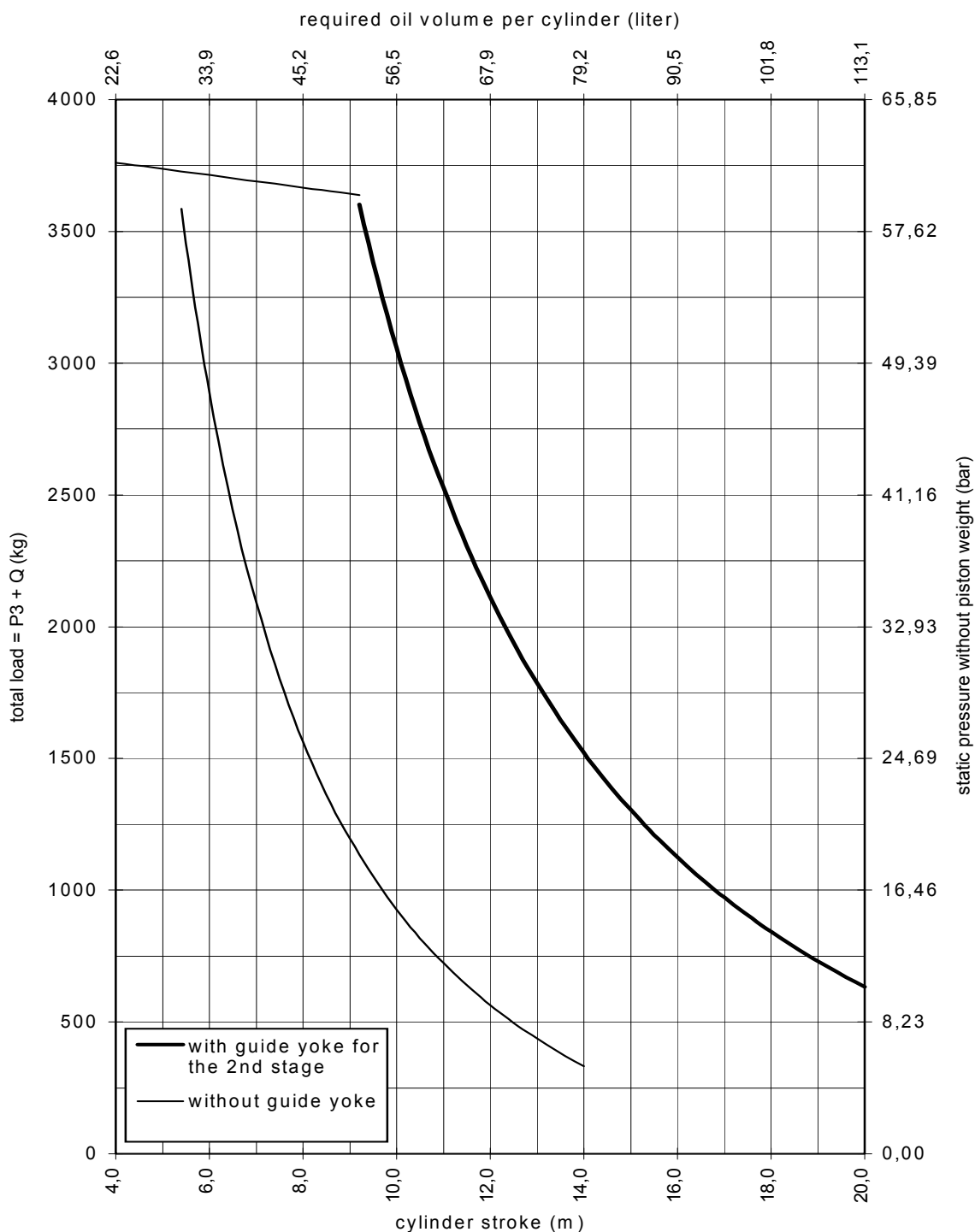


Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 63/2 - VE



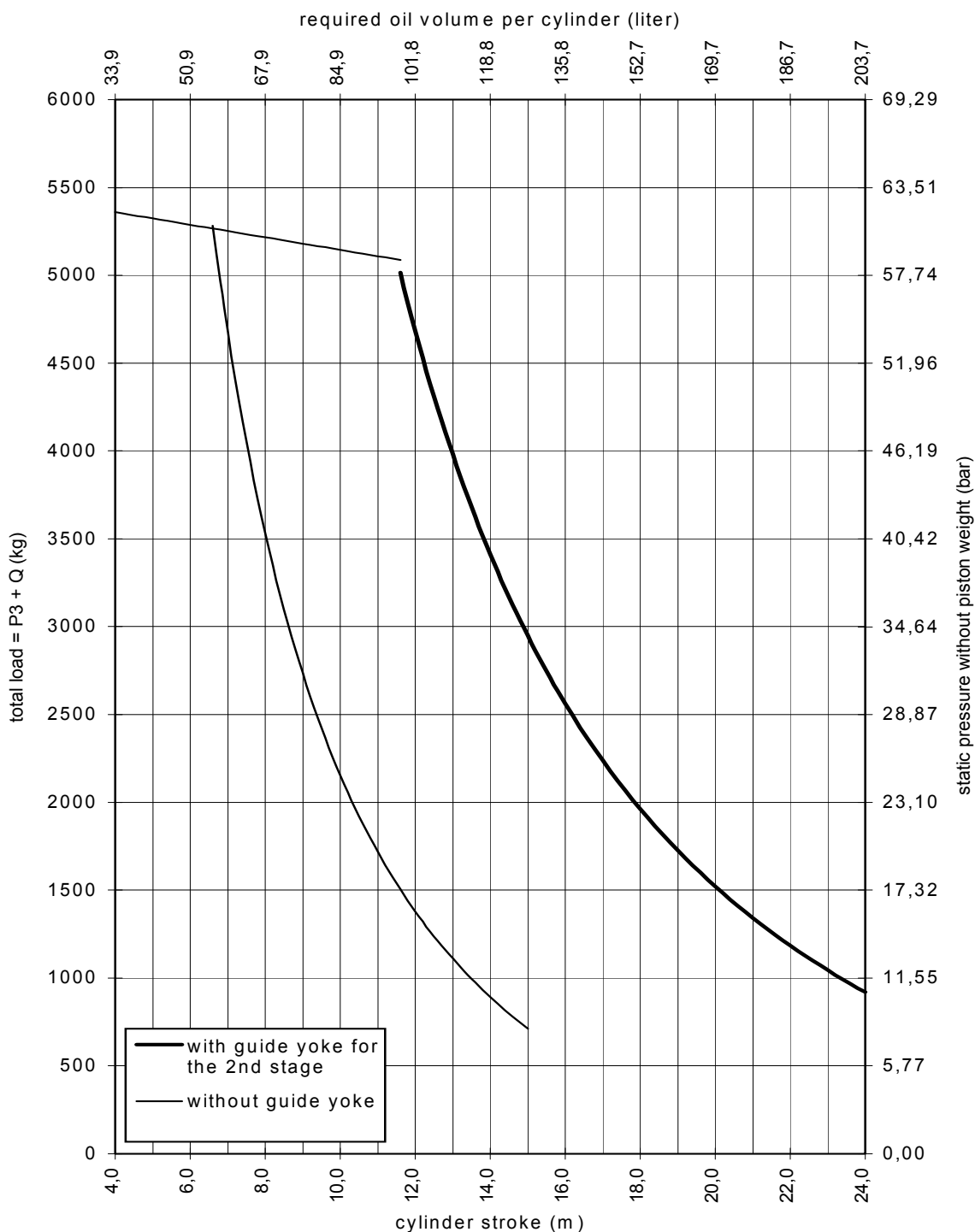
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 63 / 85 (mm)	Factor of excess pressure = 1,4
reference area	A = 43,891 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 27,602 (kg) (0 stroke) + 18,527 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 70/2 - VE



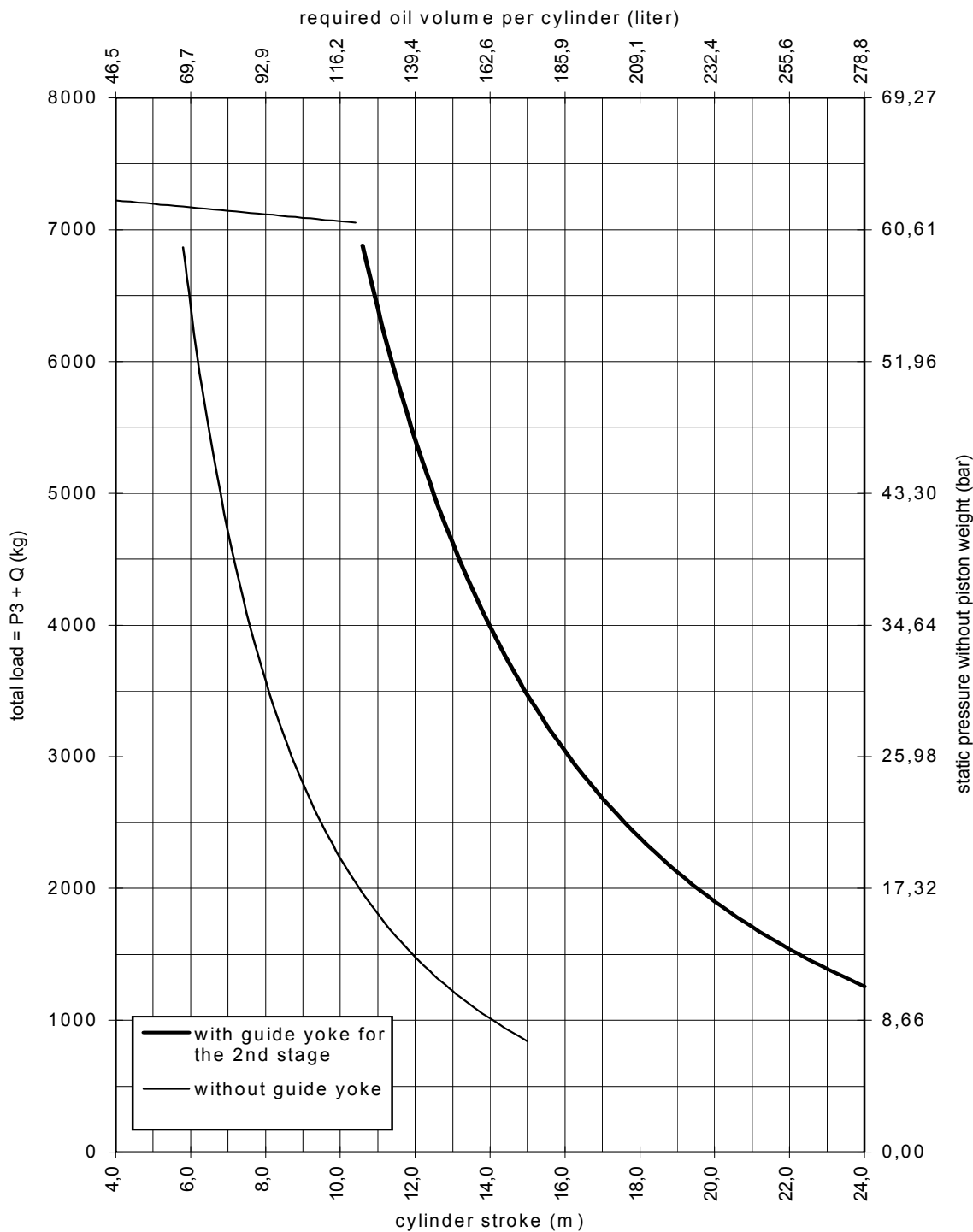
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 70 / 100 (mm)	Factor of excess pressure = 1,4
reference area	A = 59,589 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 40,184 (kg) (0 stroke) + 23,659 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 85/2 - VE



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	$p = 64$ (bar)	Factor of safety to buckling = 2,0
piston rod diameter	$d_a = 85 / 120$ (mm)	Factor of excess pressure = 1,4
reference area	$A = 84,952$ (cm <sup>2</sup> )	$Q =$ pay load (kg)
weight of piston	$m_p = 51,451$ (kg) (0 stroke) + 35,836 (kg) (per meter stroke)	$P_3 =$ weight of cabina (kg)
		$z =$ number of cylinders

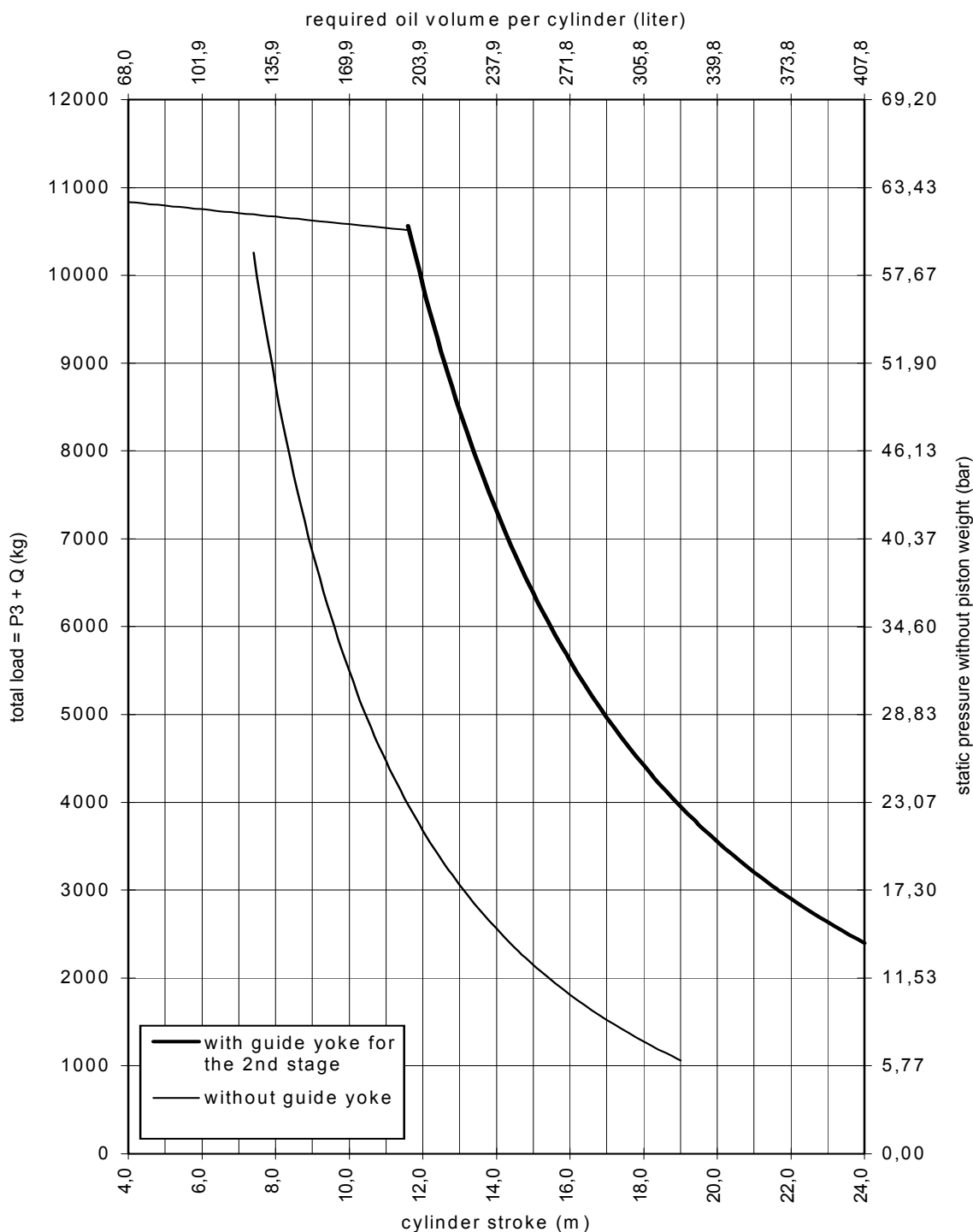
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 100/2 - VE



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 100 / 140 (mm)	Factor of excess pressure = 1,4
reference area	A = 116,269 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 70,812 (kg) (0 stroke) + 27,128 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

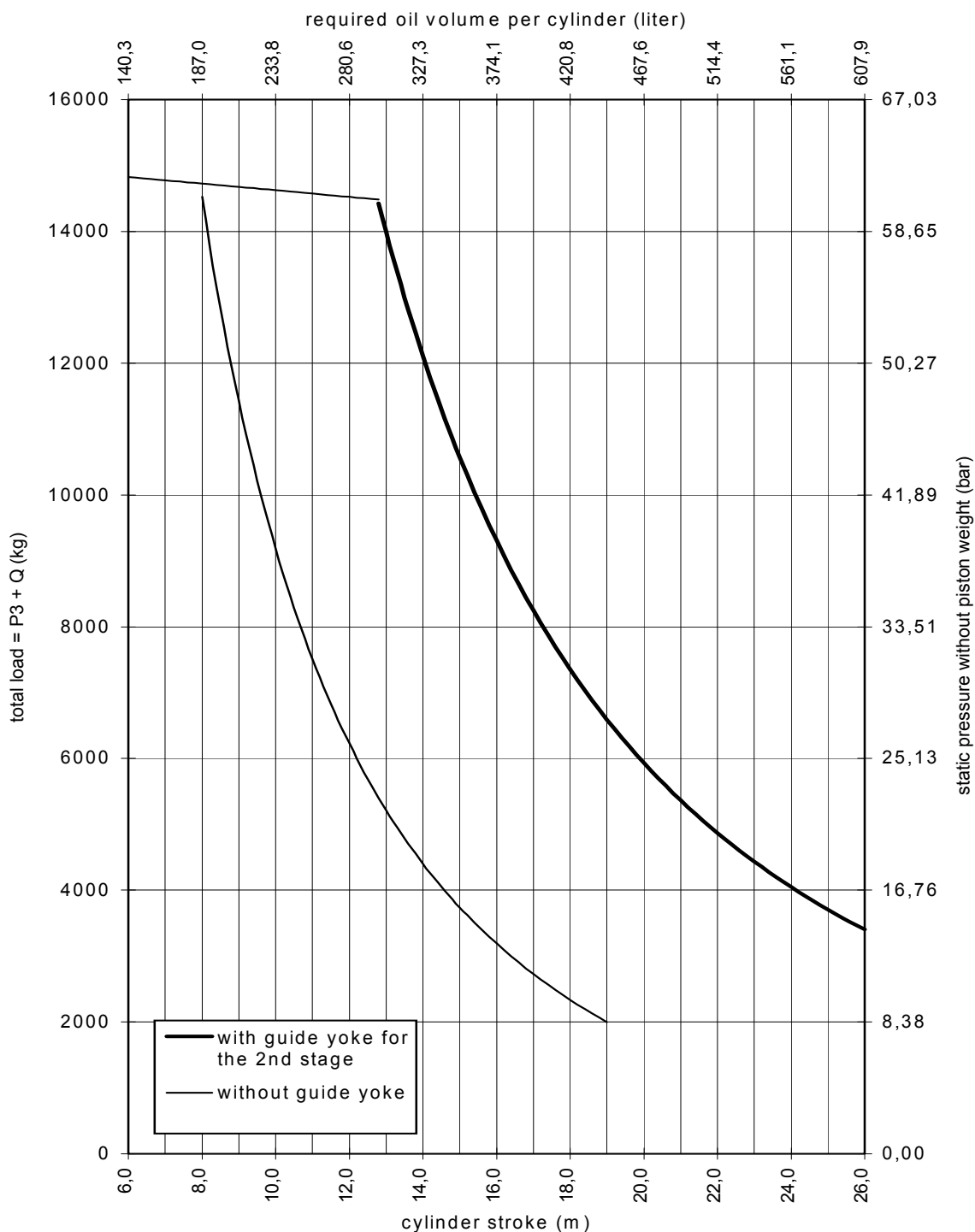


Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 120/2 - VE



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 120 / 170 (mm)	Factor of excess pressure = 1,4
reference area	A = 170,110 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 100,705 (kg) (0 stroke) + 42,233 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

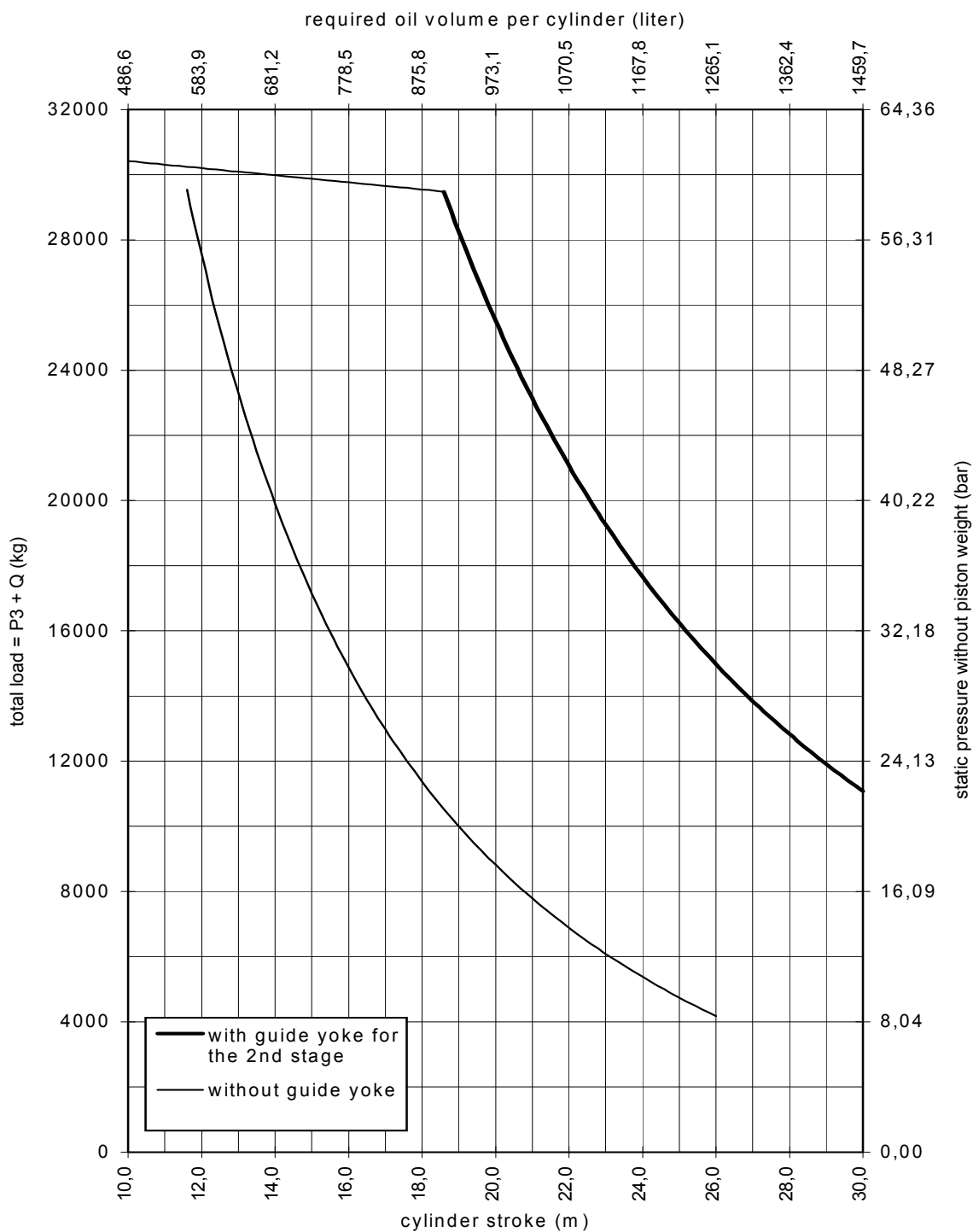
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 140/2 - VE



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 140 / 200 (mm)	Factor of excess pressure = 1,4
reference area	A = 234,180 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 159,651 (kg) (0 stroke) + 50,248 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

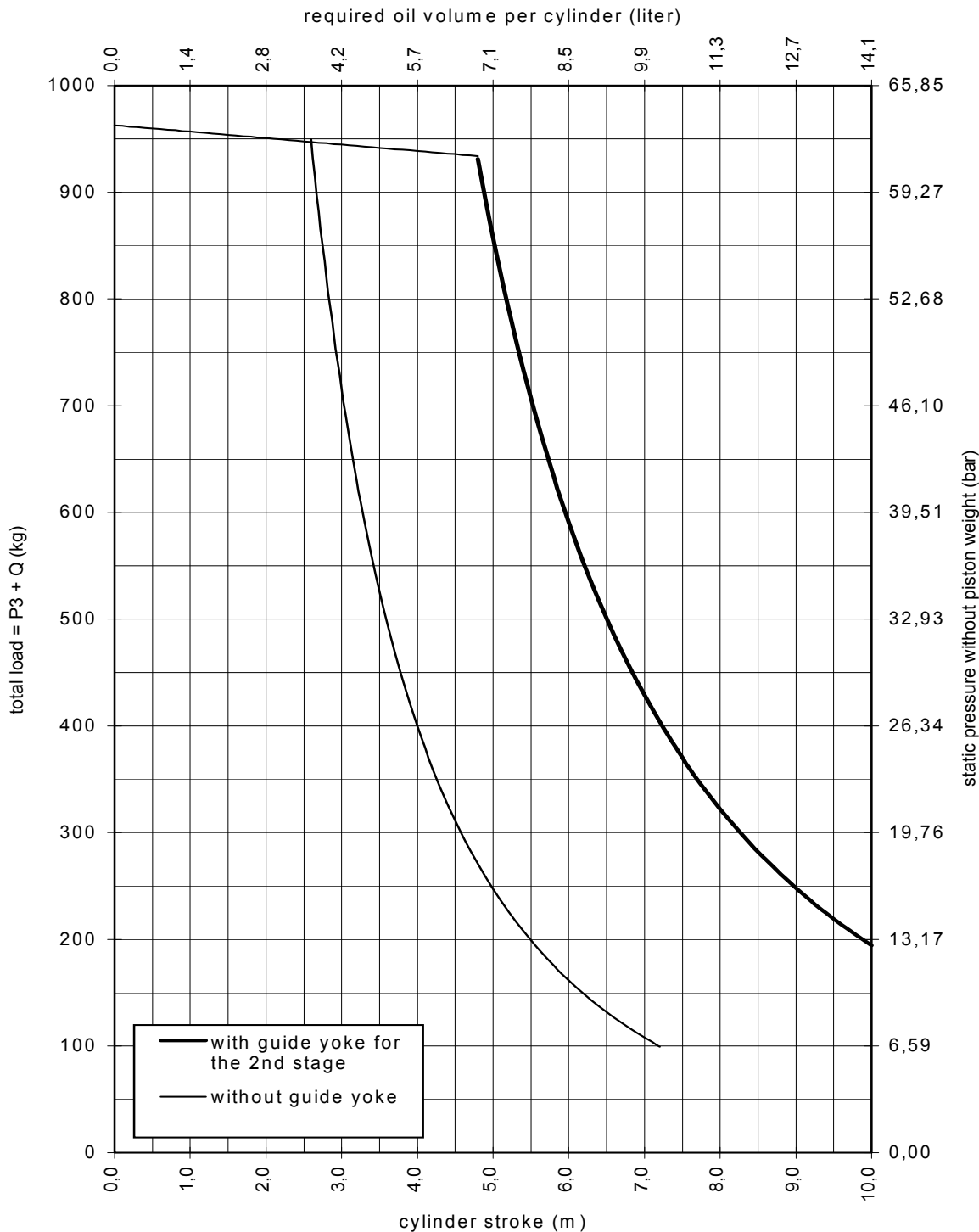


Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 200/2 - VE



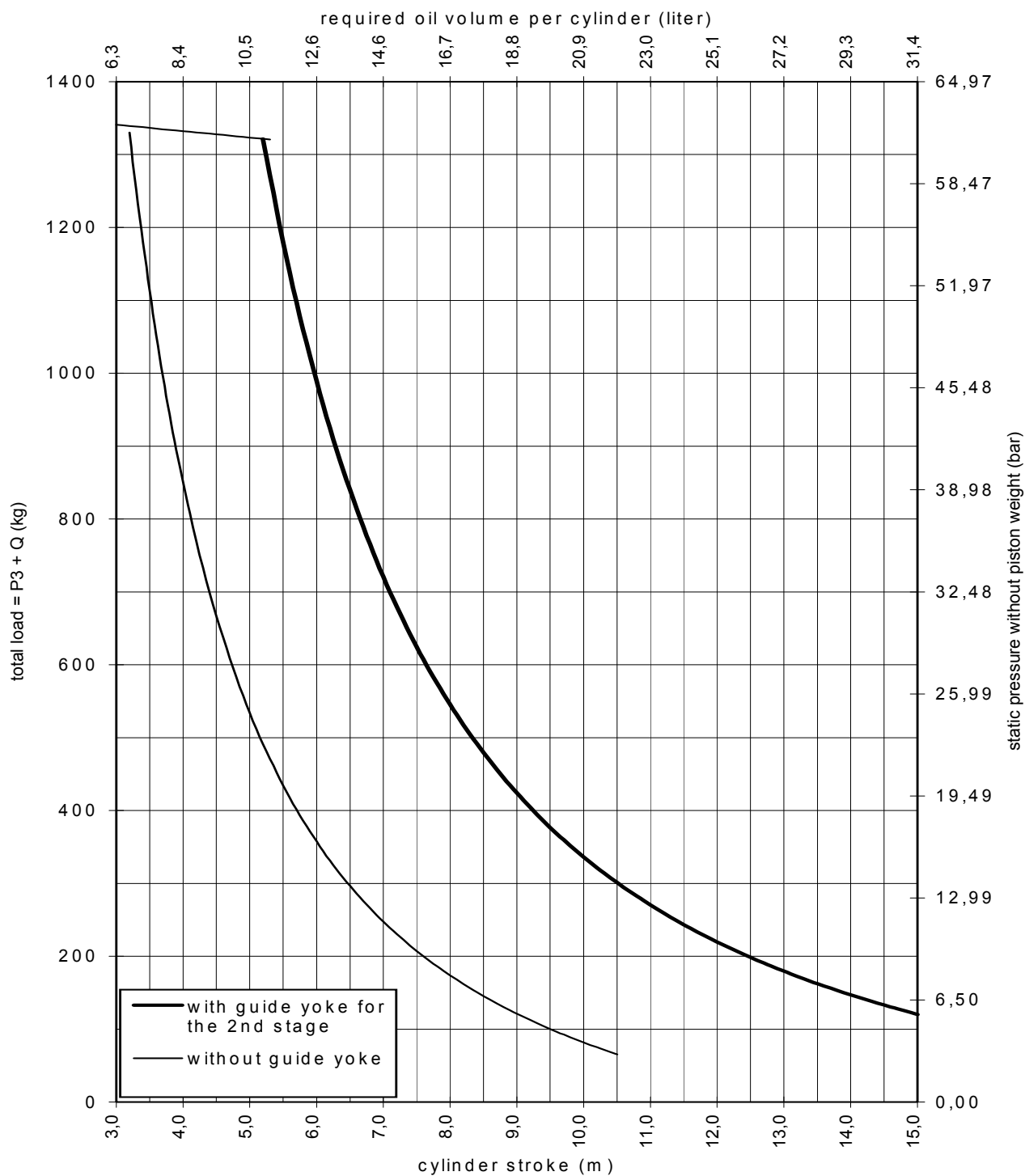
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 200 / 290 (mm)	Factor of excess pressure = 1,4
reference area	A = 487,765 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 332,140 (kg) (0 stroke) + 108,890 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 35/2 - RS



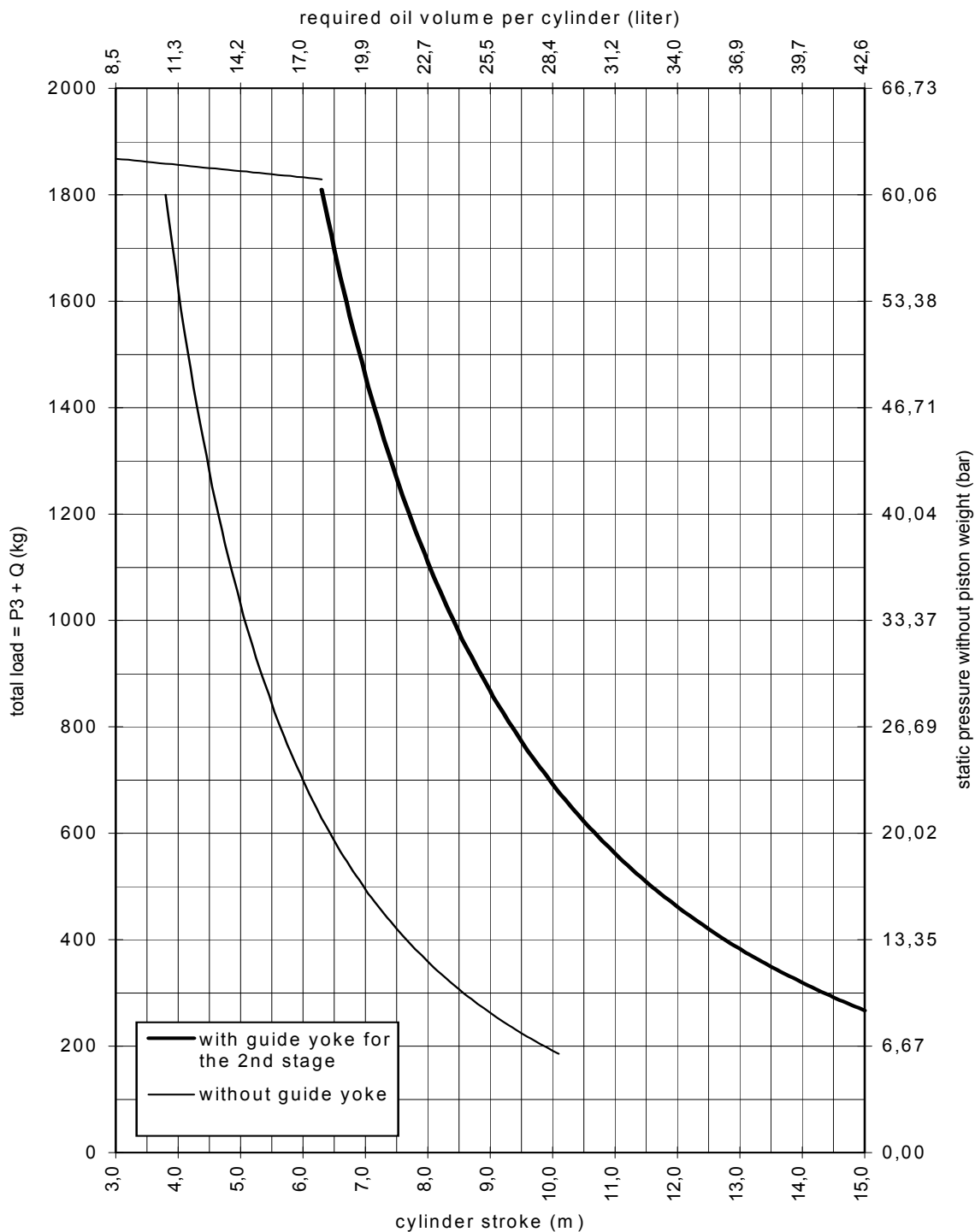
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 35 / 50 (mm)	Factor of excess pressure = 1,4
reference area	A = 14,897 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 9,175 (kg) (0 stroke) + 6,045 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 42/2 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 42 / 60 (mm)	Factor of excess pressure = 1,4
reference area	A = 21,138 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 11,7 (kg) (0 stroke) + 8,8 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

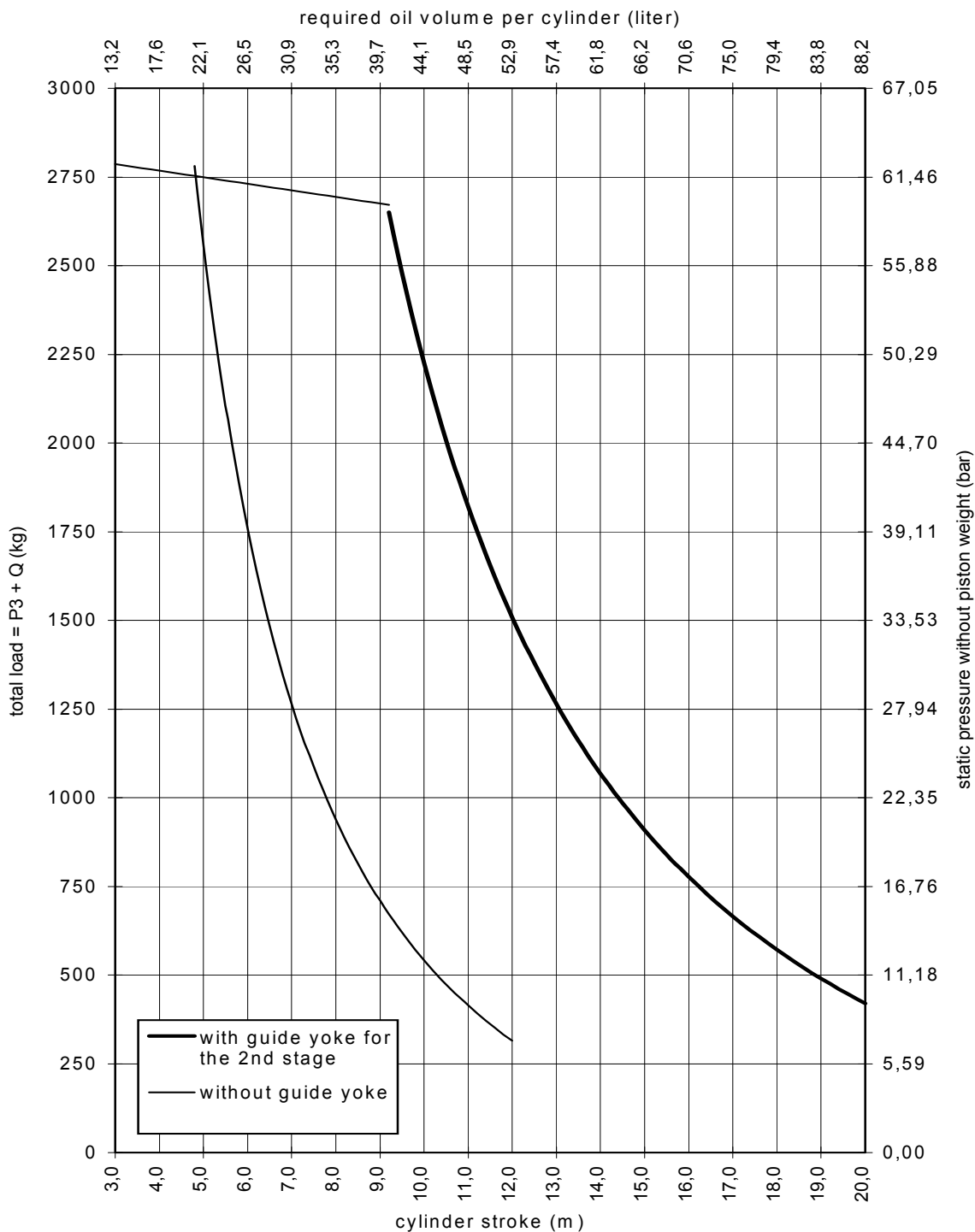
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 50/2 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 50 / 70 (mm)	Factor of excess pressure = 1,4
reference area	A = 29,402 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 15,188 (kg) (0 stroke) + 11,714 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

<b>C.O.A.M.</b> S.p.A. COMPONENTI OLEODINAMICI PER ASCENSORI E MONTACARICHI	Selection diagram for telescopic 2-stage cylinder - Side Ram System	emesso	S. A.	20/11/1998
		controllato		02/07/2002
		nullaosta		
Sost. il	2 PX 0271a	DOCUMENTAZIONI TECNICHE 2 P X 0 2 7 1 b		
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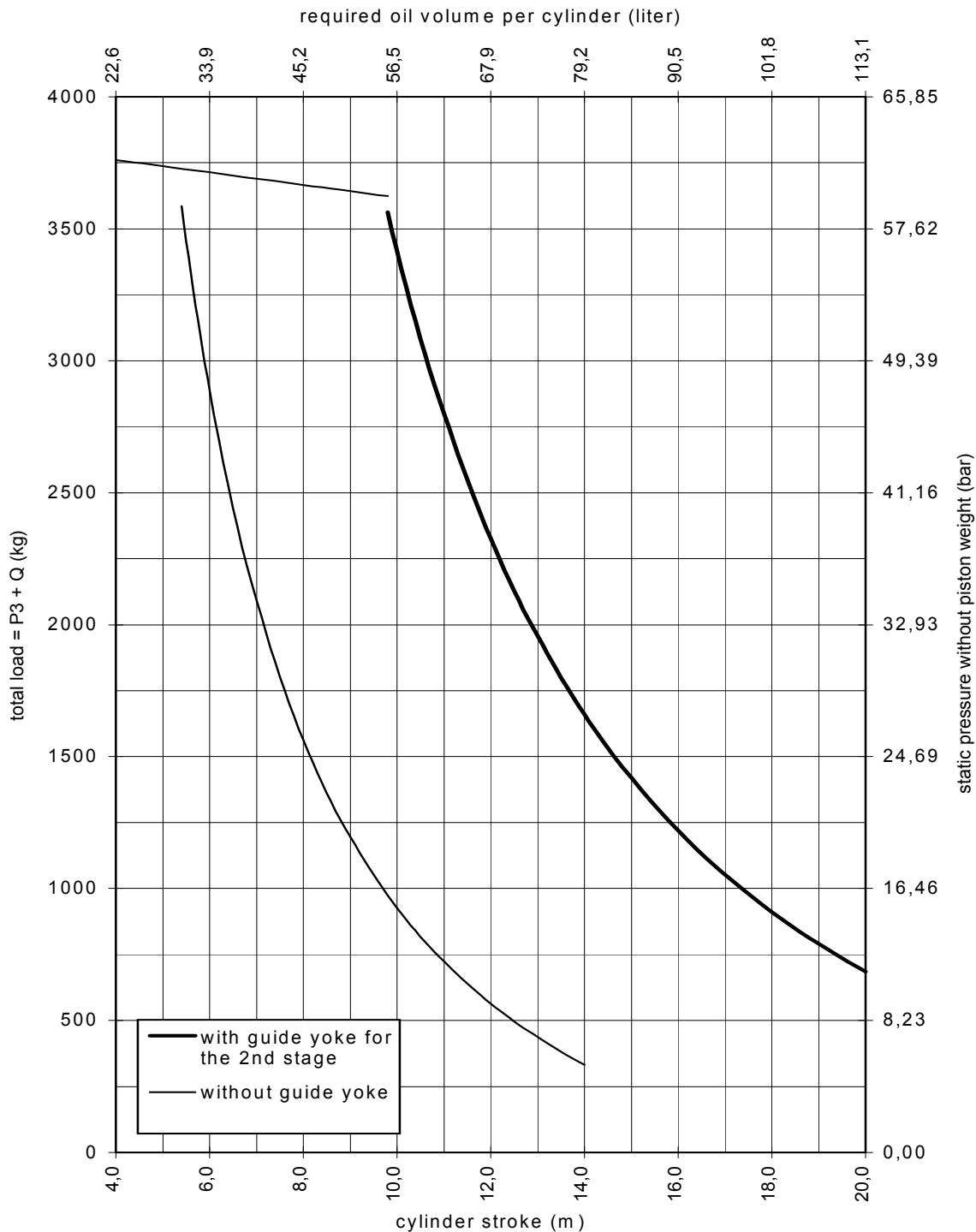
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 63/2 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 63 / 85 (mm)	Factor of excess pressure = 1,4
reference area	A = 43,891 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 20,897 (kg) (0 stroke) + 18,527 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders



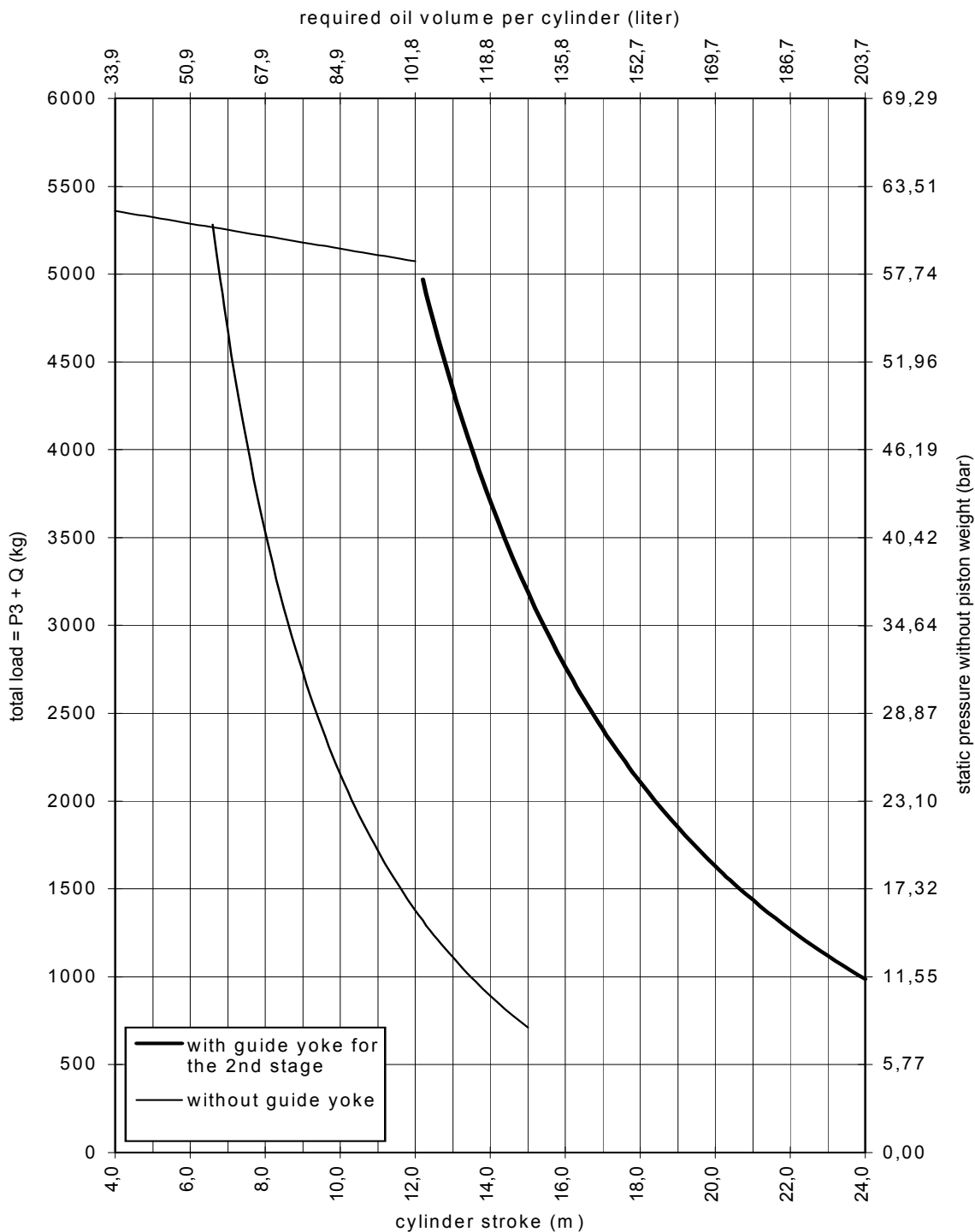
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 70/2 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 70 / 100 (mm)	Factor of excess pressure = 1,4
reference area	A = 59,589 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 31,906 (kg) (0 stroke) + 23,659 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

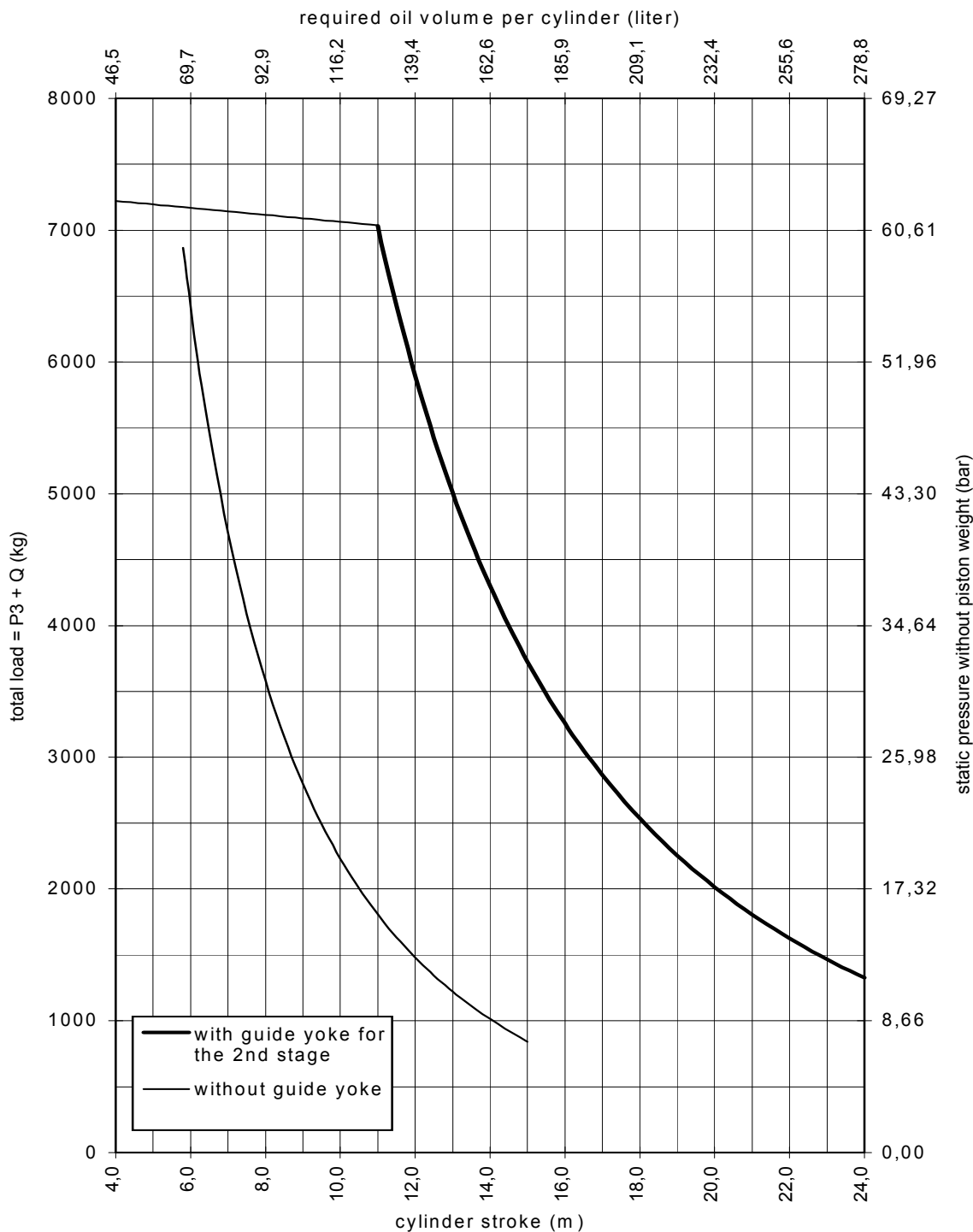
<b>C.O.A.M.</b> S.p.A. COMPONENTI OLEODINAMICI PER ASCENSORI E MONTACARICHI	Selection diagram for telescopic 2-stage cylinder - Side Ram System	emesso	S. A.	20/11/1998
		controllato		02/07/2002
		nullaosta		
Sost. il	2 PX 0271a	DOCUMENTAZIONI TECNICHE 2 P X 0 2 7 1 b		
Sost. da				

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 85/2 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 85 / 120 (mm)	Factor of excess pressure = 1,4
reference area	A = 84,952 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 39,246 (kg) (0 stroke) + 35,836 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

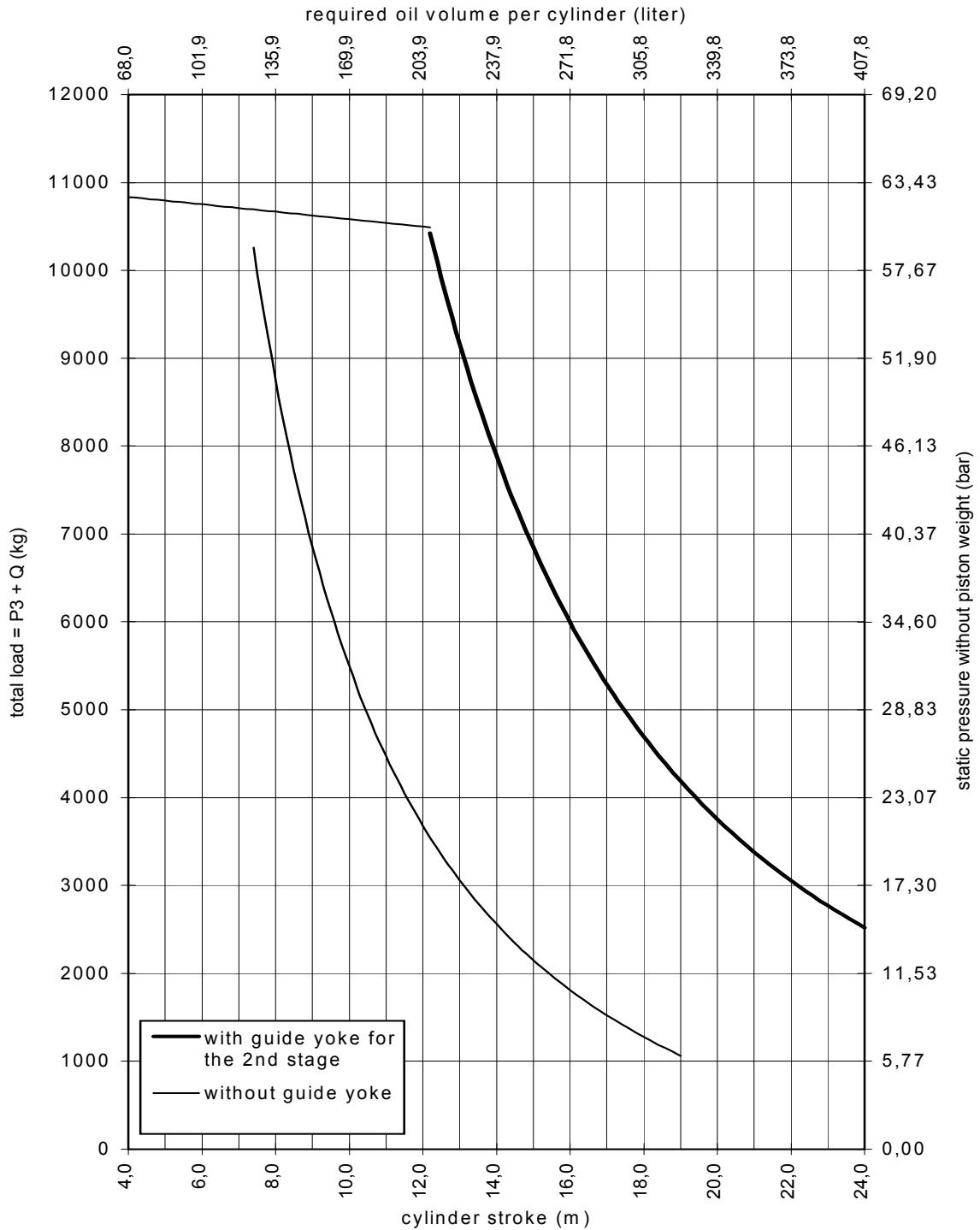
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 100/2 - RS



Selection diagram for Telescopic 2-stage Cylinder

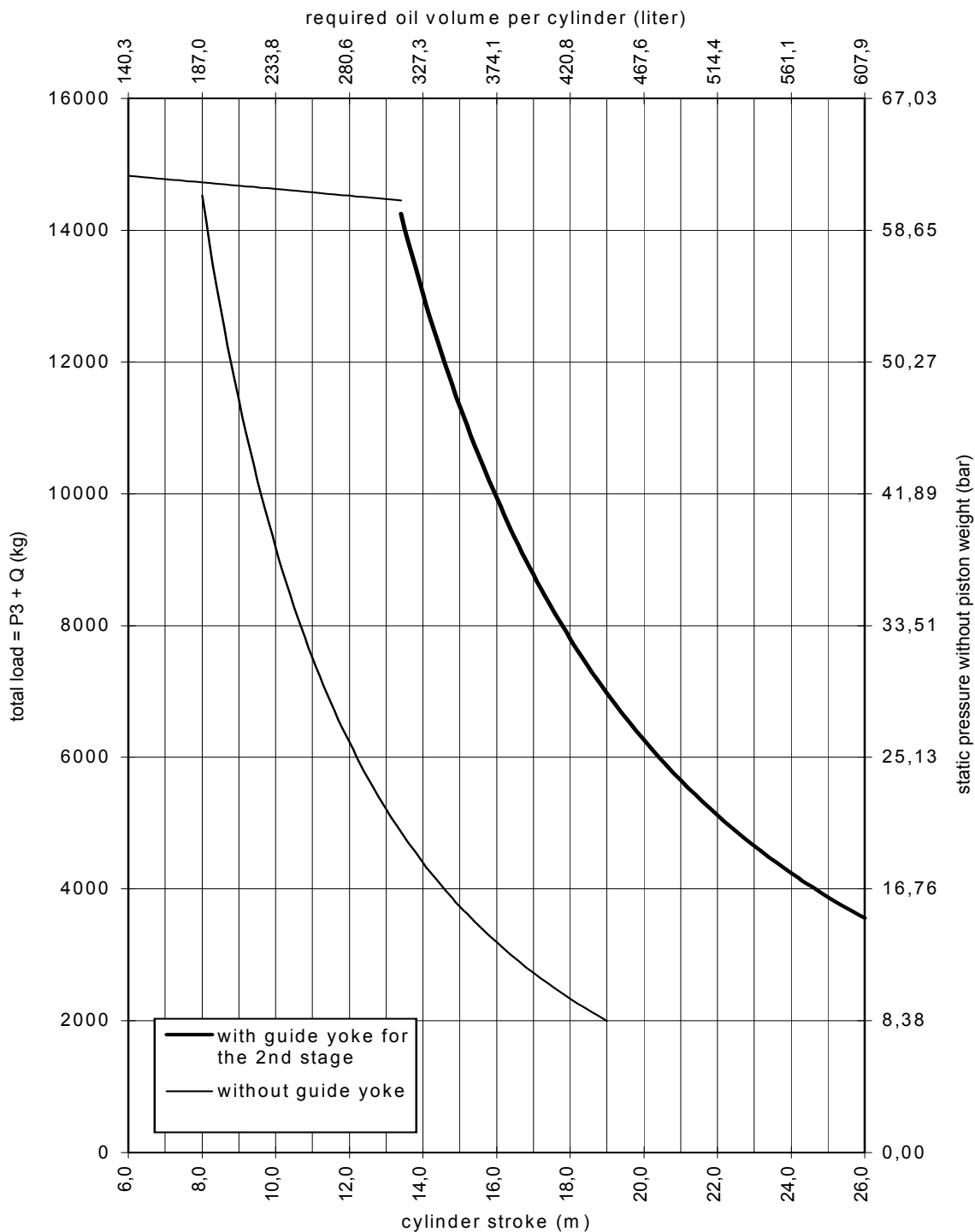
Technical data :	$p_{stat} = 0,981 \cdot \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 100 / 140 (mm)	Factor of excess pressure = 1,4
reference area	A = 116,269 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 64,730 (kg) (0 stroke) + 27,128 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Acc. to EN 81/2 - Side Ram System  
Type 3PL 120/2 - RS



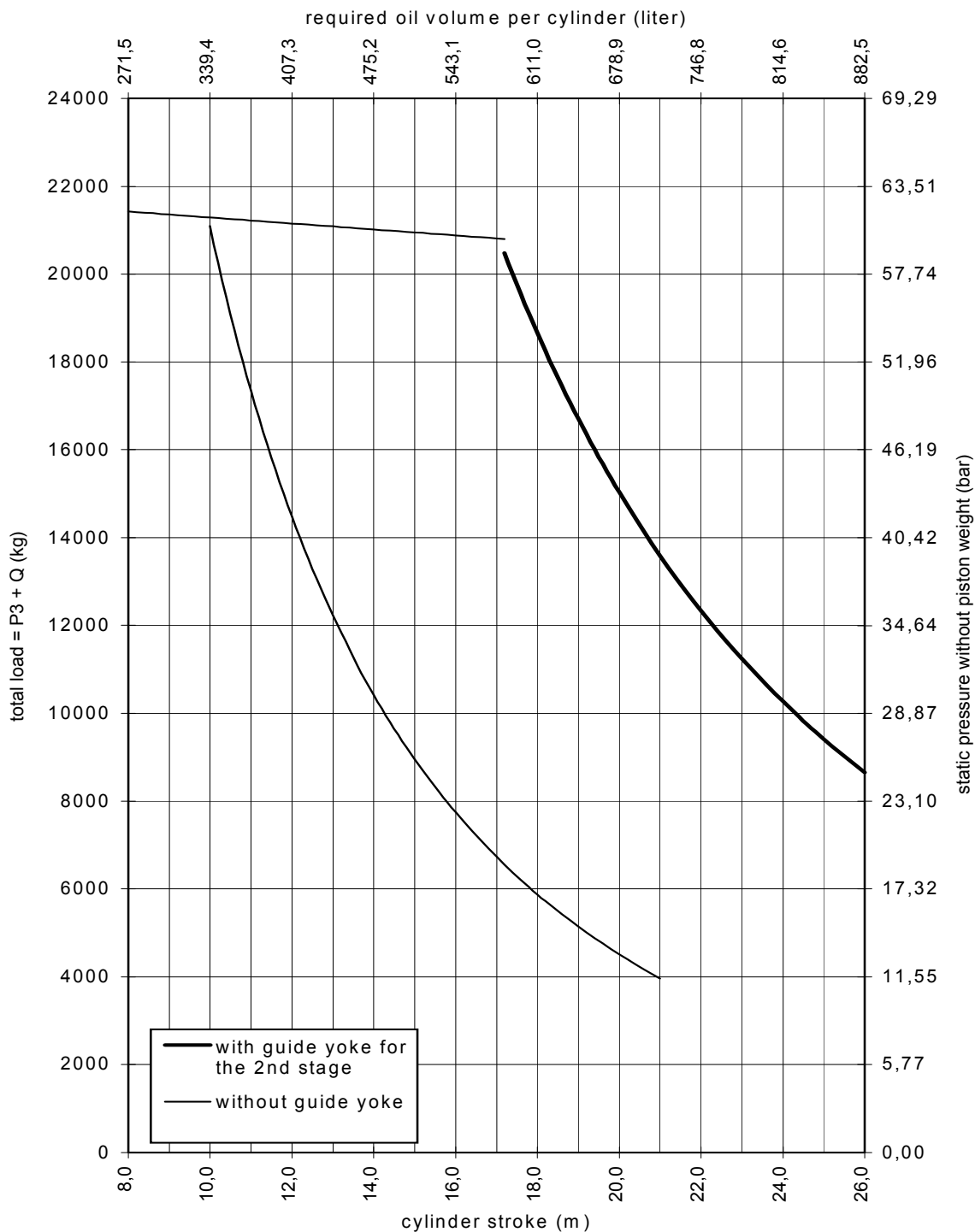
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 140 / 200 (mm)	Factor of excess pressure = 1,4
reference area	A = 234,180 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 150,867 (kg) (0 stroke) + 50,248 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 140/2 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 140 / 200 (mm)	Factor of excess pressure = 1,4
reference area	A = 234,180 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 150,867 (kg) (0 stroke) + 50,248 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

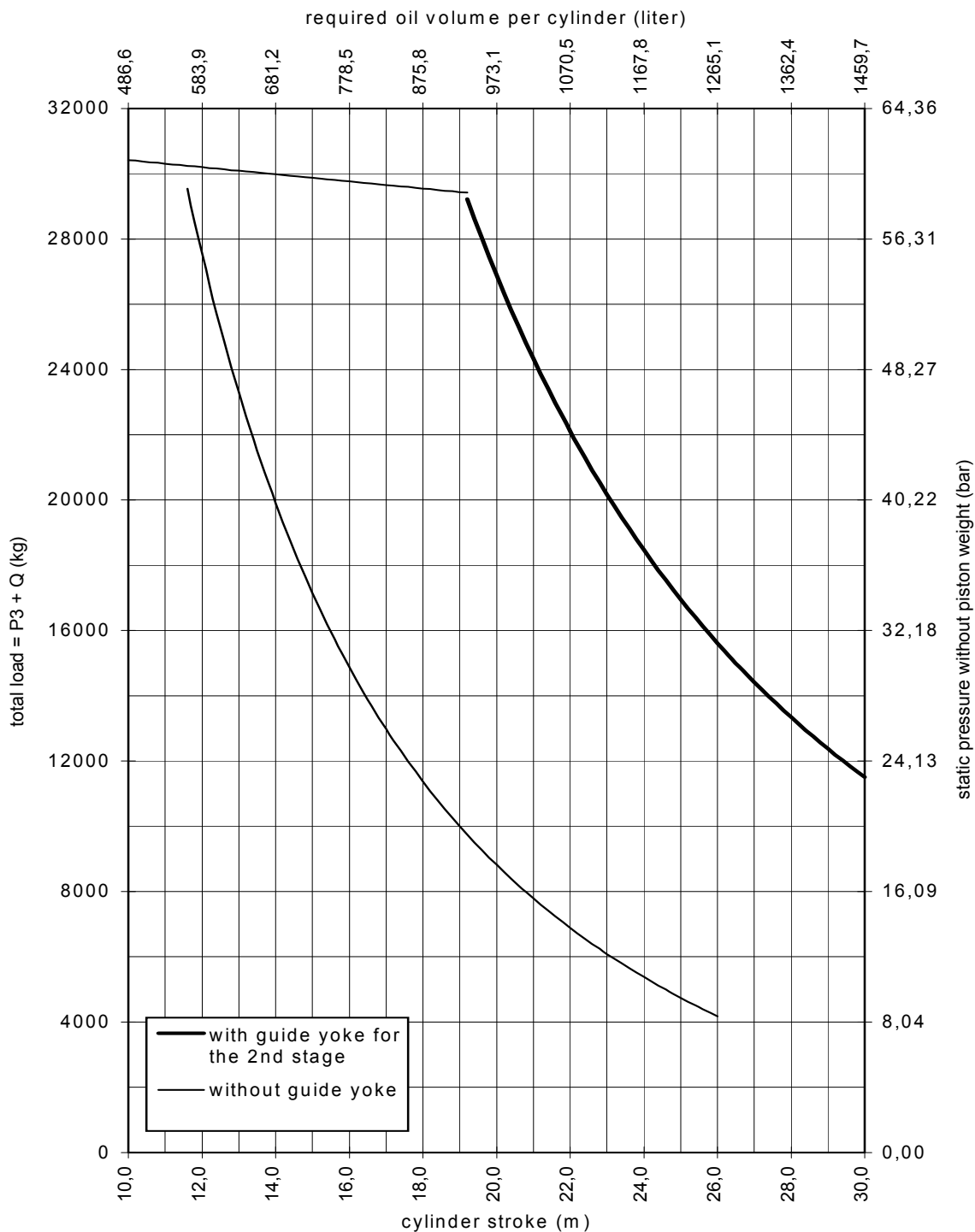
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 170/2 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 170 / 240 (mm)	Factor of excess pressure = 1,4
reference area	A = 339,810 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 202,835 (kg) (0 stroke) + 67,844 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

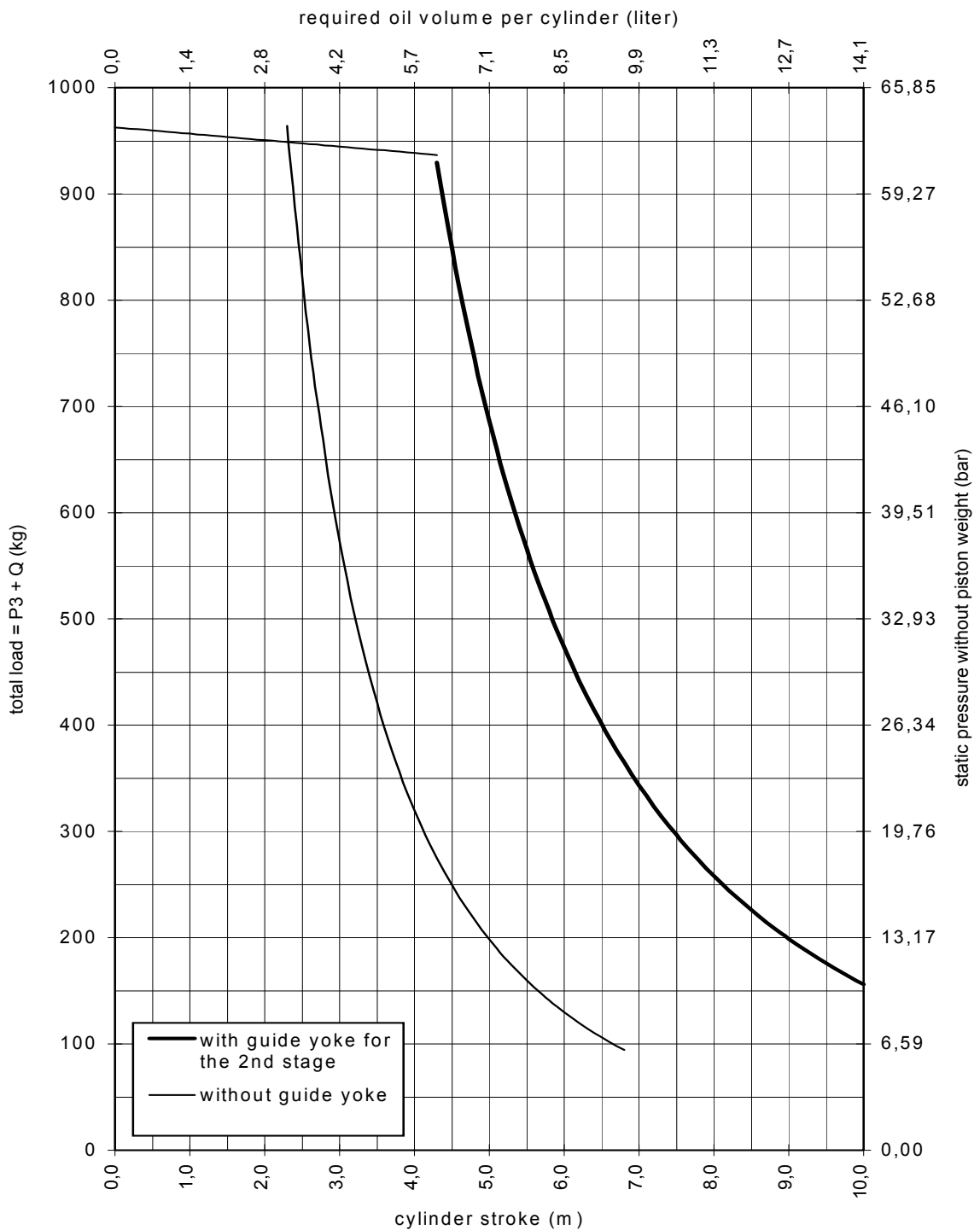
<b>C.O.A.M.</b> S.p.A. COMPONENTI OLEODINAMICI PER ASCENSORI E MONTACARICHI	Selection diagram for telescopic 2-stage cylinder - Side Ram System		emesso	S. A.	20/11/1998
			controllato		02/07/2002
			nullaosta		
Sost. il	2 PX 0271a	DOCUMENTAZIONI TECNICHE	2 P X 0 2 7 1 b	pag	10
Sost. da				di	11

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 200/2 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 200 / 290 (mm)	Factor of excess pressure = 1,4
reference area	A = 487,765 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 313,965 (kg) (0 stroke) + 108,890 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

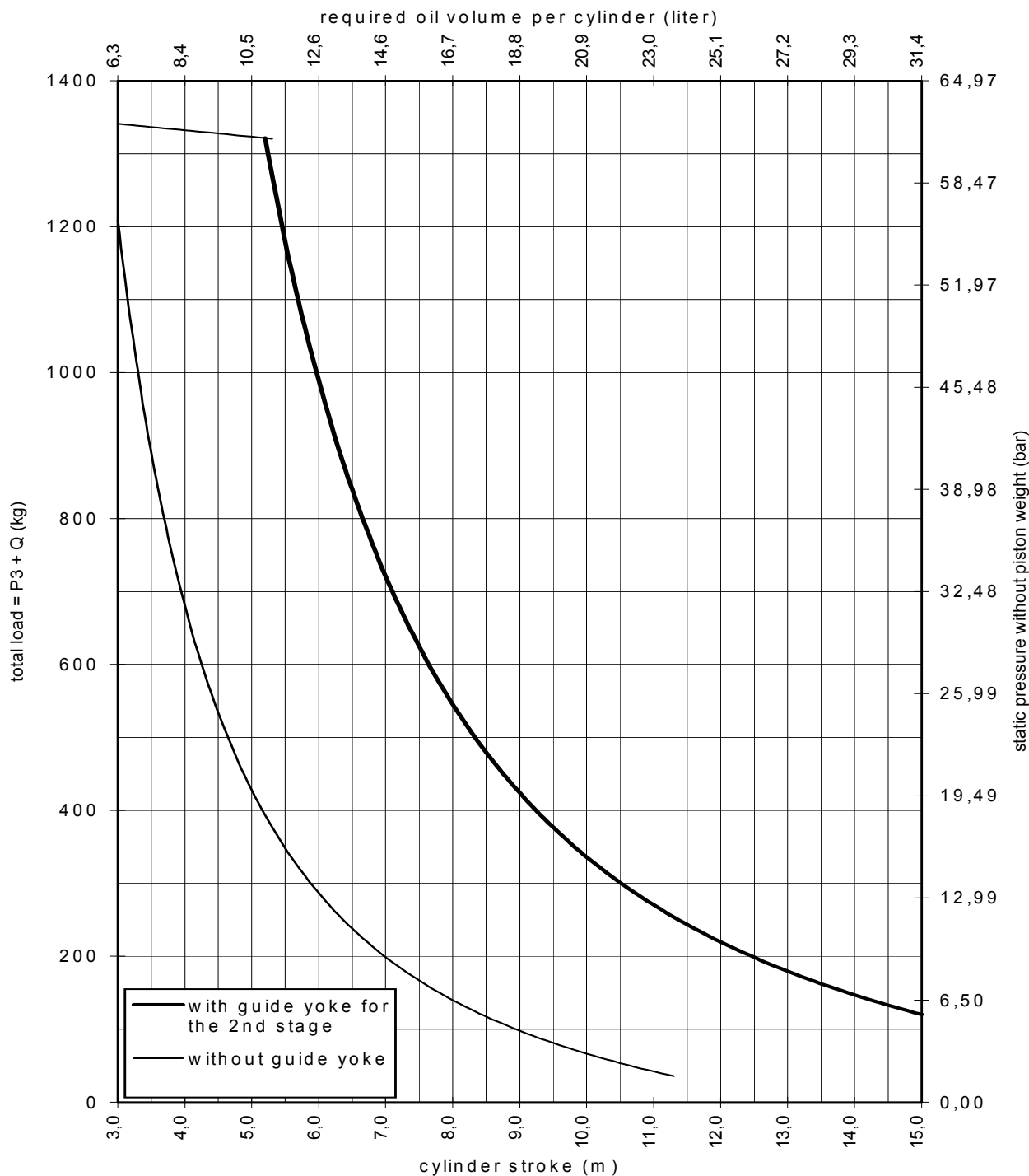
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 35/2 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 35 / 50 (mm)	Factor of excess pressure = 1,4
reference area	A = 14,897 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 9,175 (kg) (0 stroke) + 6,045 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

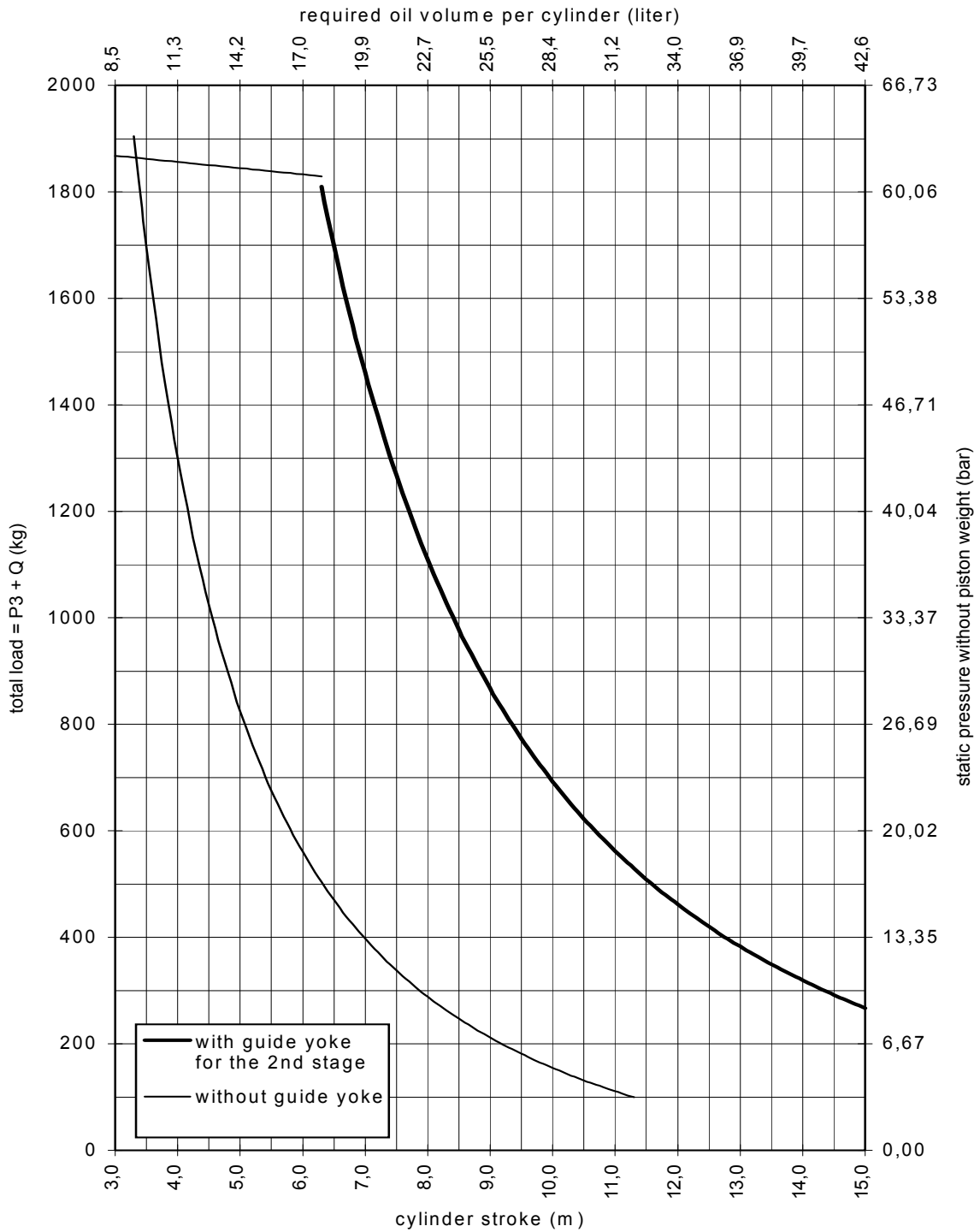


Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 42/2 - VT



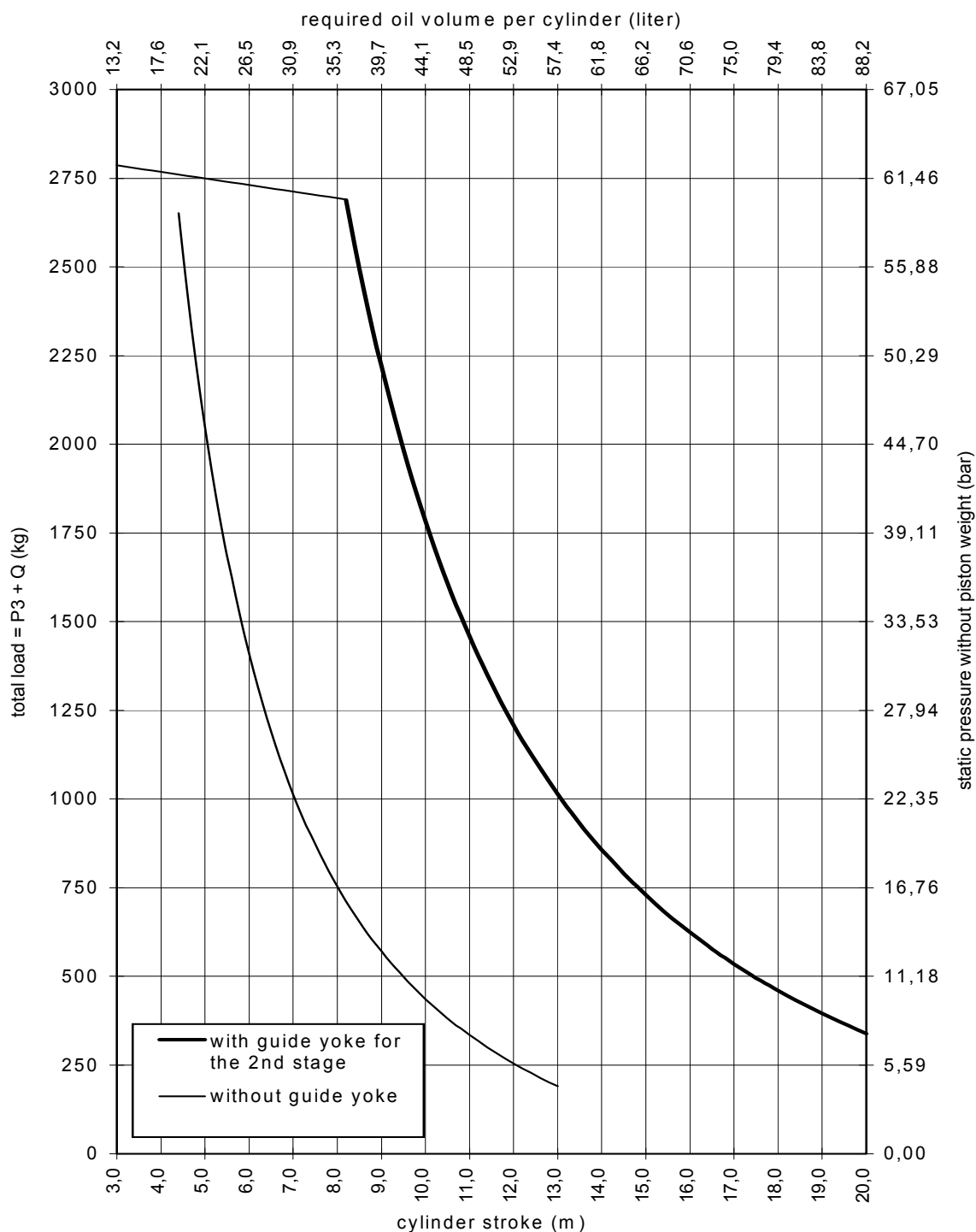
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 42 / 60 (mm)	Factor of excess pressure = 1,4
reference area	A = 21,138 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 11,7 (kg) (0 stroke) + 8,8 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 50/2 - VT



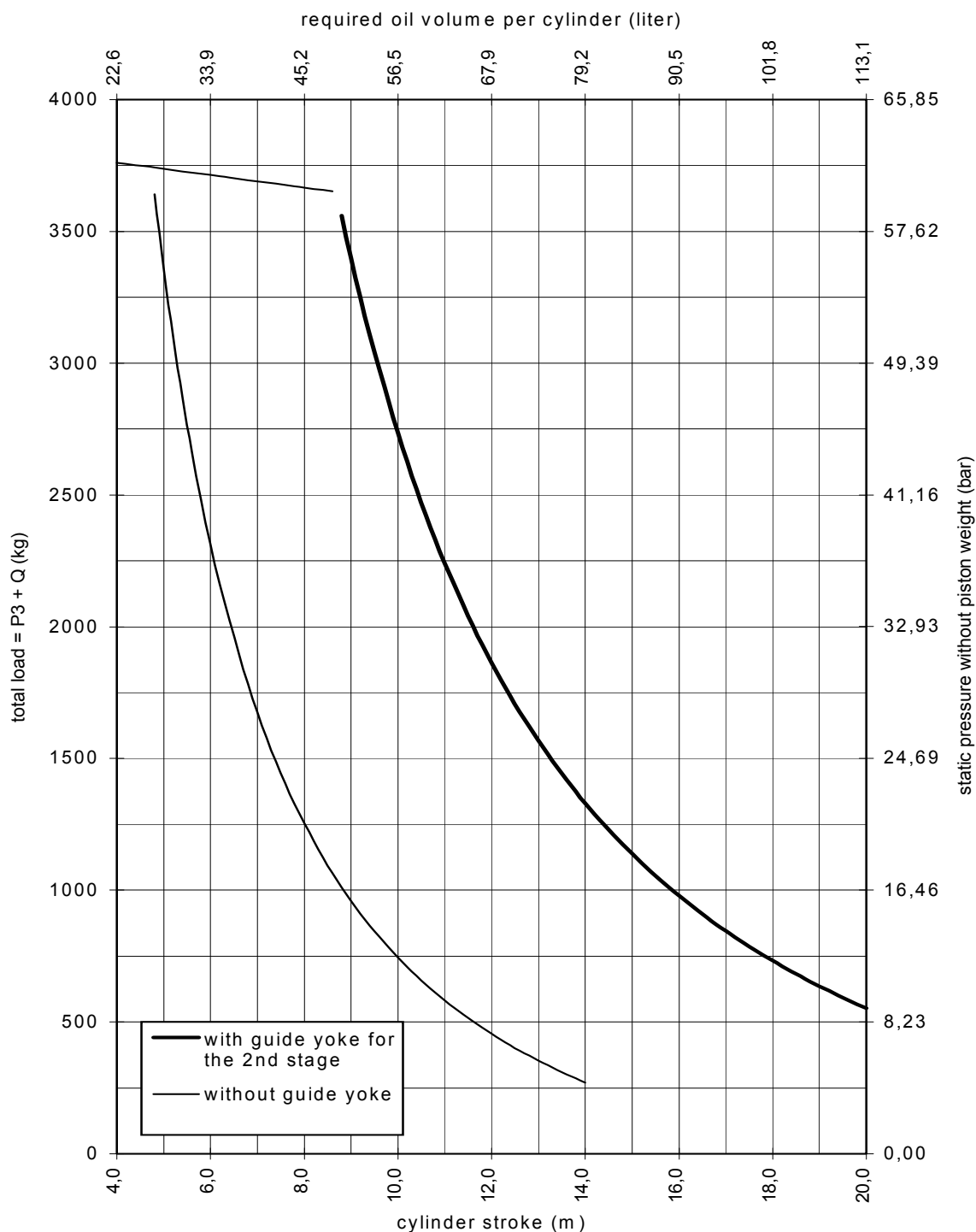
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	$p = 64$ (bar)	Factor of safety to buckling = 2,5
piston rod diameter	$d_a = 50 / 70$ (mm)	Factor of excess pressure = 1,4
reference area	$A = 29,402$ (cm <sup>2</sup> )	$Q =$ pay load (kg)
weight of piston	$m_p = 15,188$ (kg) (0 stroke) + 11,714 (kg) (per meter stroke)	$P_3 =$ weight of cabina (kg) $z =$ number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 63/2 - VT



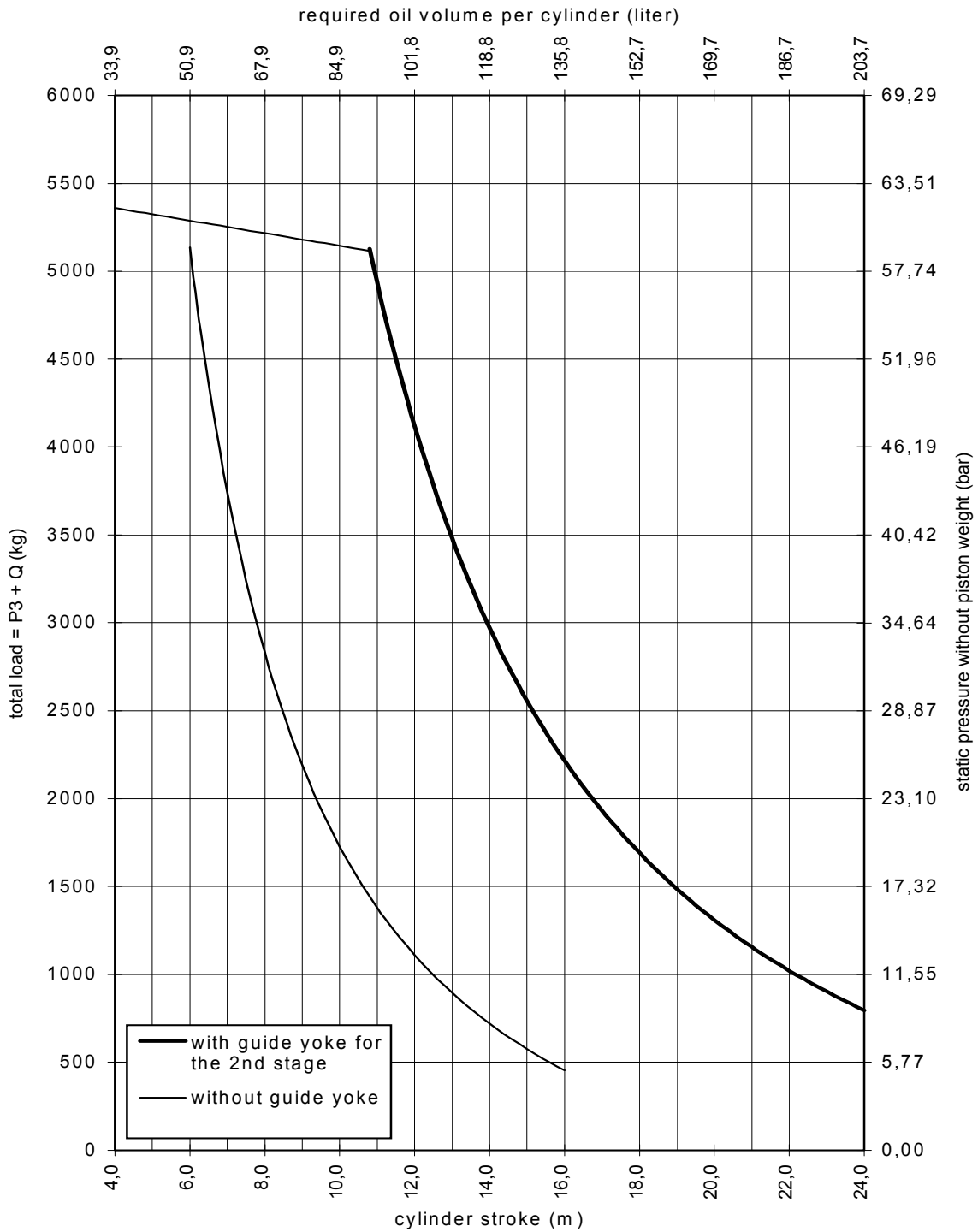
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 63 / 85 (mm)	Factor of excess pressure = 1,4
reference area	A = 43,891 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 20,897 (kg) (0 stroke) + 18,527 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 70/2 - VT



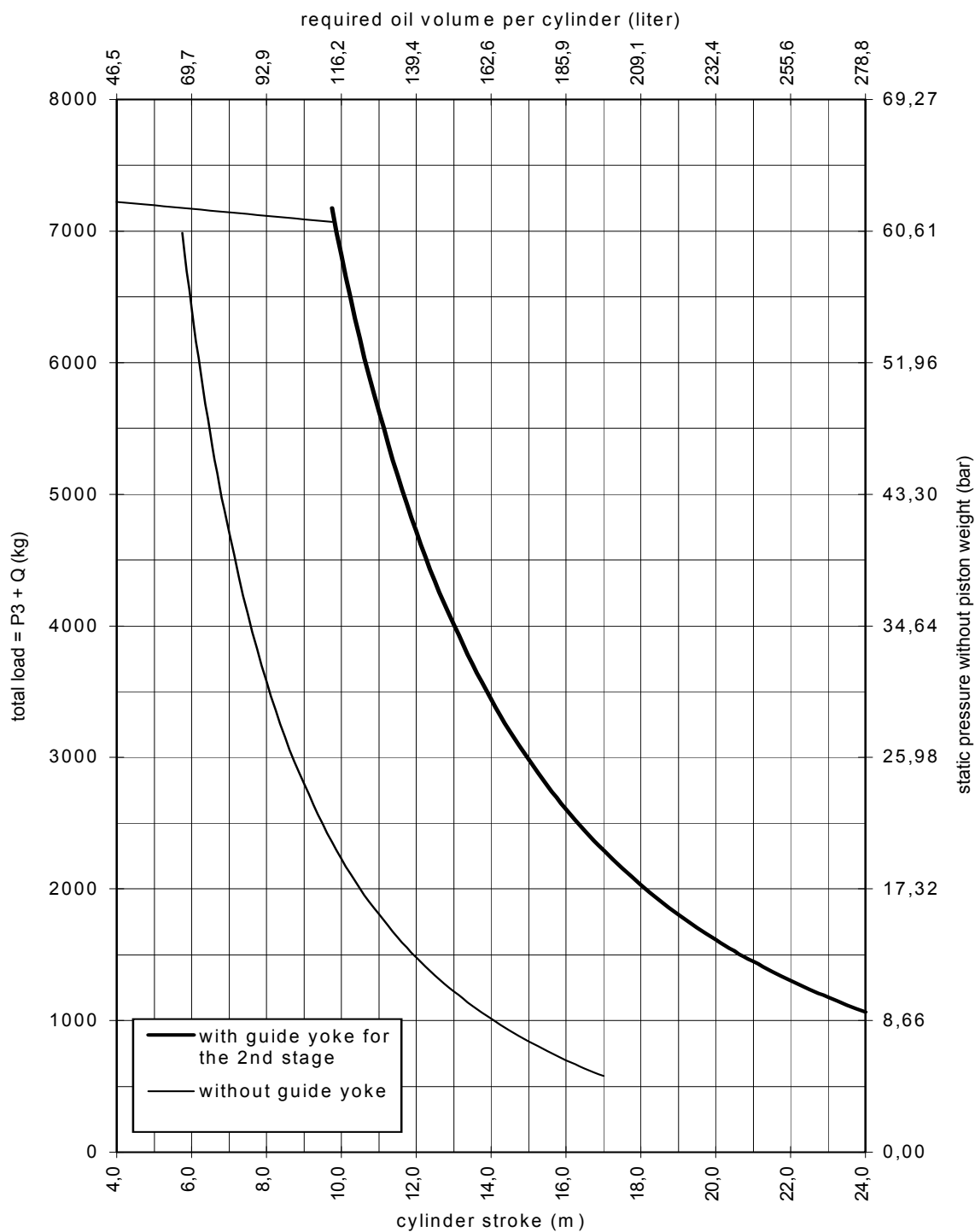
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 70 / 100 (mm)	Factor of excess pressure = 1,4
reference area	A = 59,589 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 31,906 (kg) (0 stroke) + 23,659 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 85/2 - VT



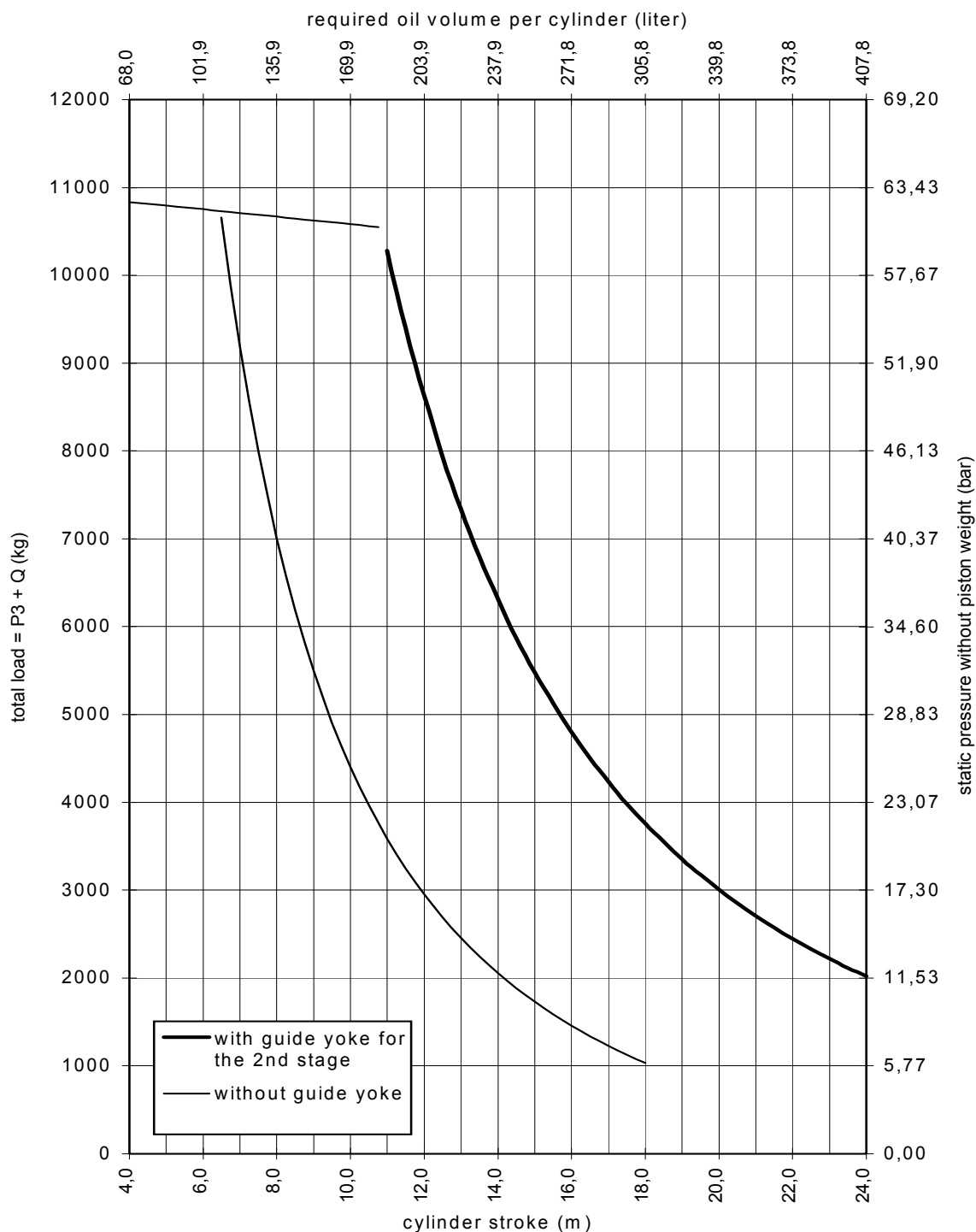
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 85 / 120 (mm)	Factor of excess pressure = 1,4
reference area	A = 84,952 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 39,246 (kg) (0 stroke) + 35,836 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 100/2 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	$p = 64$ (bar)	Factor of safety to buckling = 2,5
piston rod diameter	$d_a = 100 / 140$ (mm)	Factor of excess pressure = 1,4
reference area	$A = 116,269$ (cm <sup>2</sup> )	$Q =$ pay load (kg)
weight of piston	$m_p = 64,730$ (kg) (0 stroke) + 27,128 (kg) (per meter stroke)	$P_3 =$ weight of cabina (kg) $z =$ number of cylinders

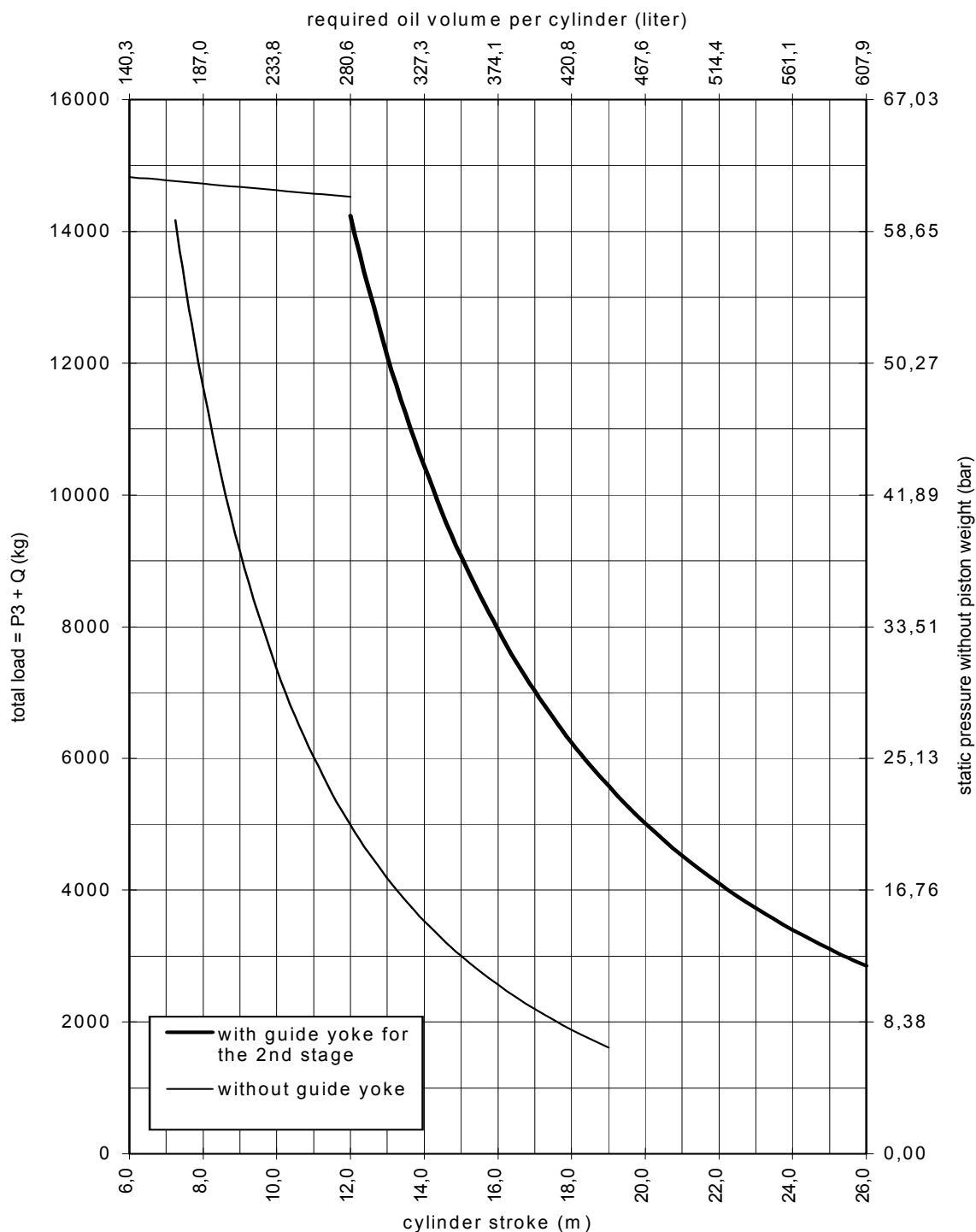
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 120/2 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 120 / 170 (mm)	Factor of excess pressure = 1,4
reference area	A = 170,110 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 93,272 (kg) (0 stroke) + 42,233 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

<b>C.O.A.M.</b> S.p.A. COMPONENTI OLEODINAMICI PER ASCENSORI E MONTACARICHI	Selection diagram for telescopic 2-stage cylinder - VT Series	emesso	S. A.	20/11/1998
		controllato		02-07-2002
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		di	11	

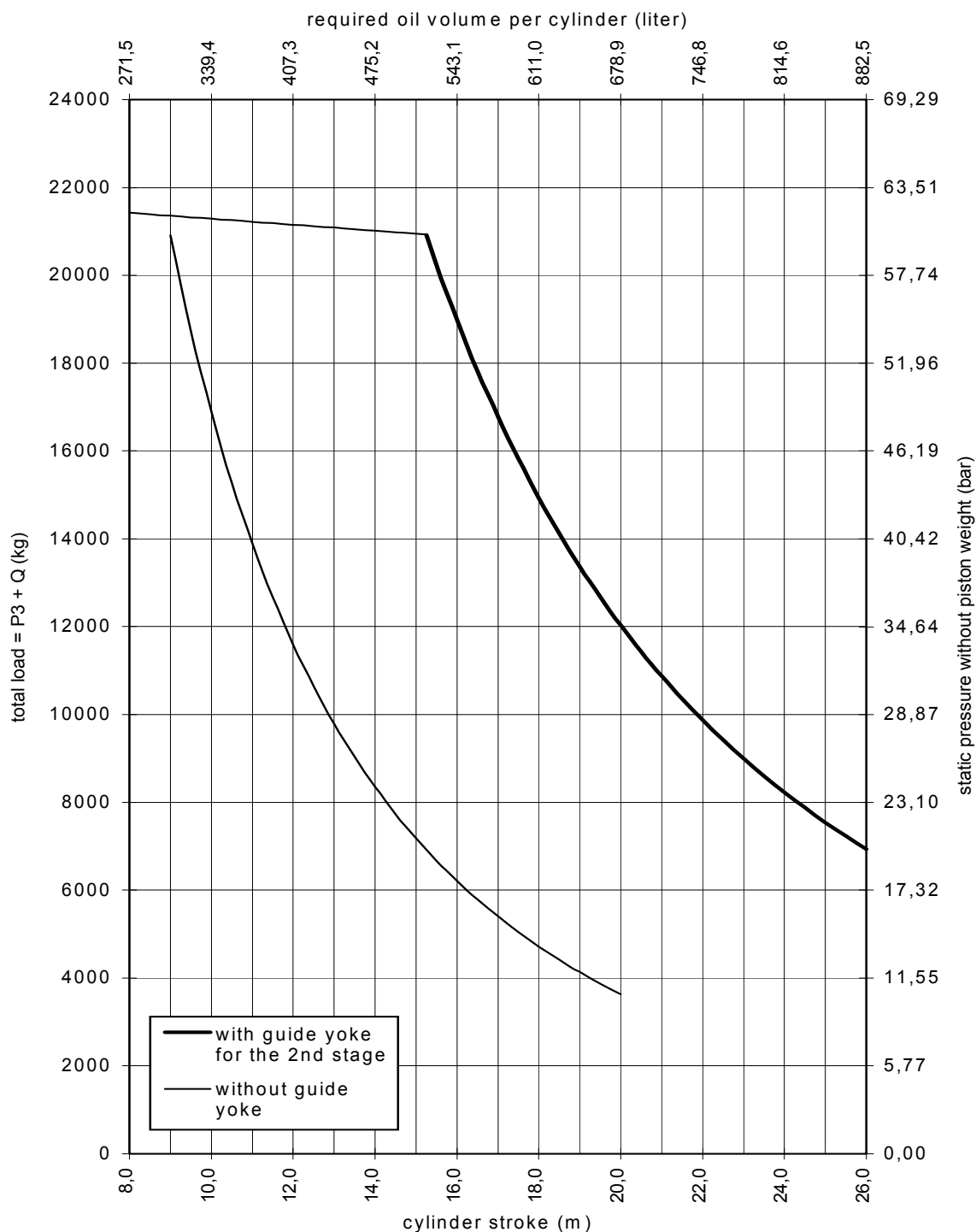
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 140/2 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 140 / 200 (mm)	Factor of excess pressure = 1,4
reference area	A = 234,180 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 150,867 (kg) (0 stroke) + 50,248 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg) z = number of cylinders

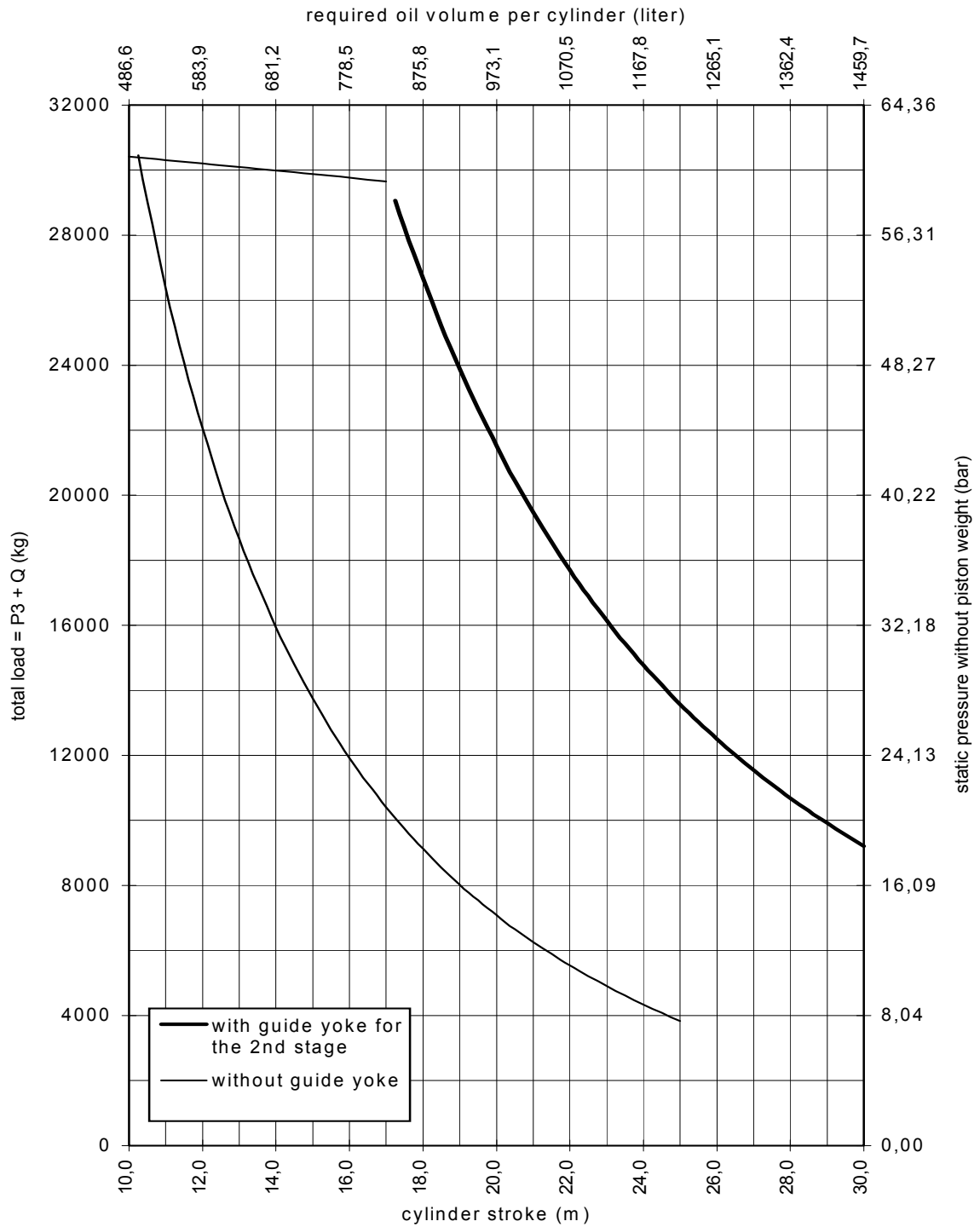


Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 170/2 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 170 / 240 (mm)	Factor of excess pressure = 1,4
reference area	A = 339,810 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 202,835 (kg) (0 stroke) + 67,844 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

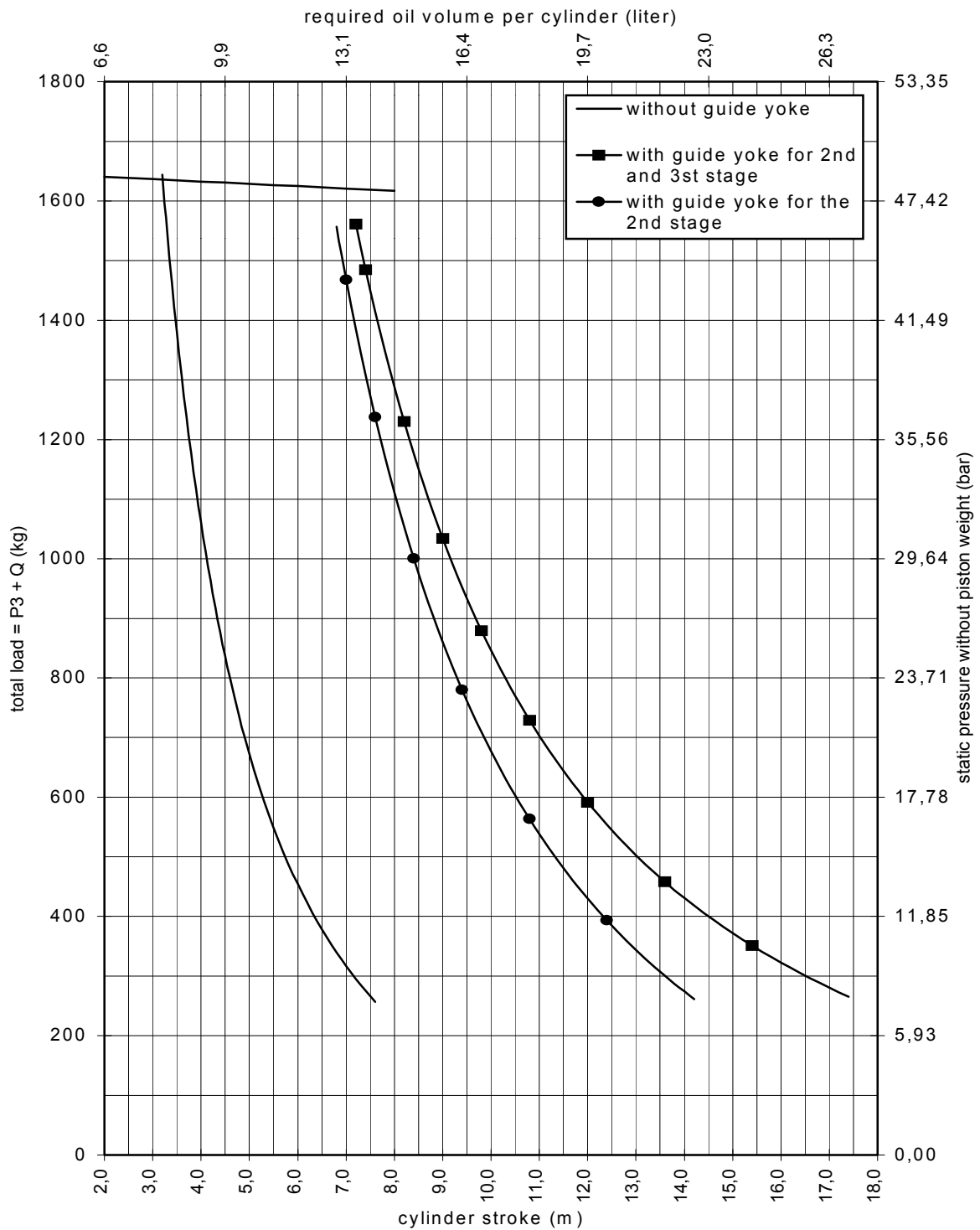
Selection diagram for Telescopic 2-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 200/2 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 64 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 200 / 290 (mm)	Factor of excess pressure = 1,4
reference area	A = 487,765 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 313,965 (kg) (0 stroke) + 108,890 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg) z = number of cylinders



Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 42/3 - VE



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 50 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 42 / 60 / 85 (mm)	Factor of excess pressure = 1,4
reference area	A = 33,100 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 32,790 (kg) (0 stroke) + 9,782 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders





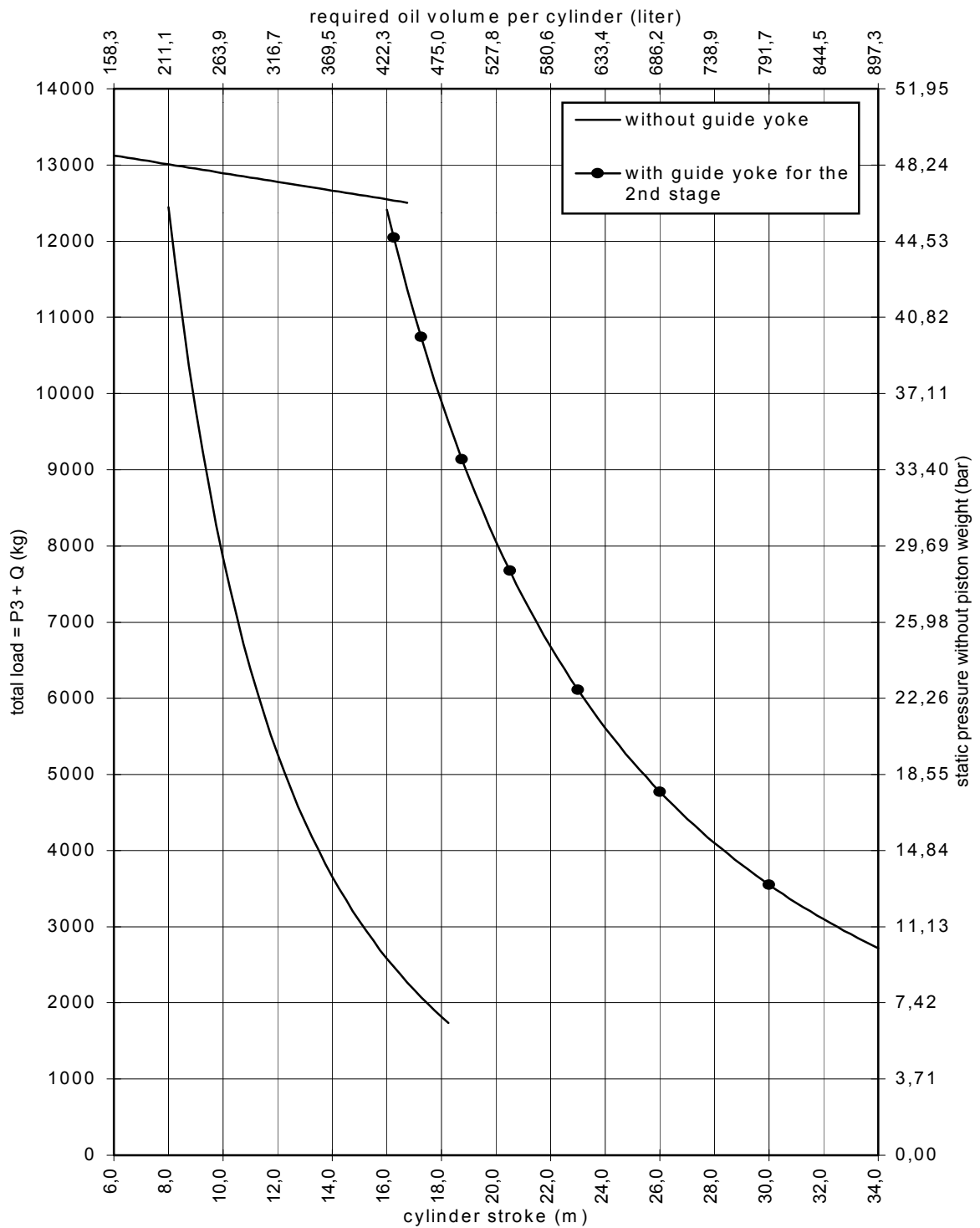






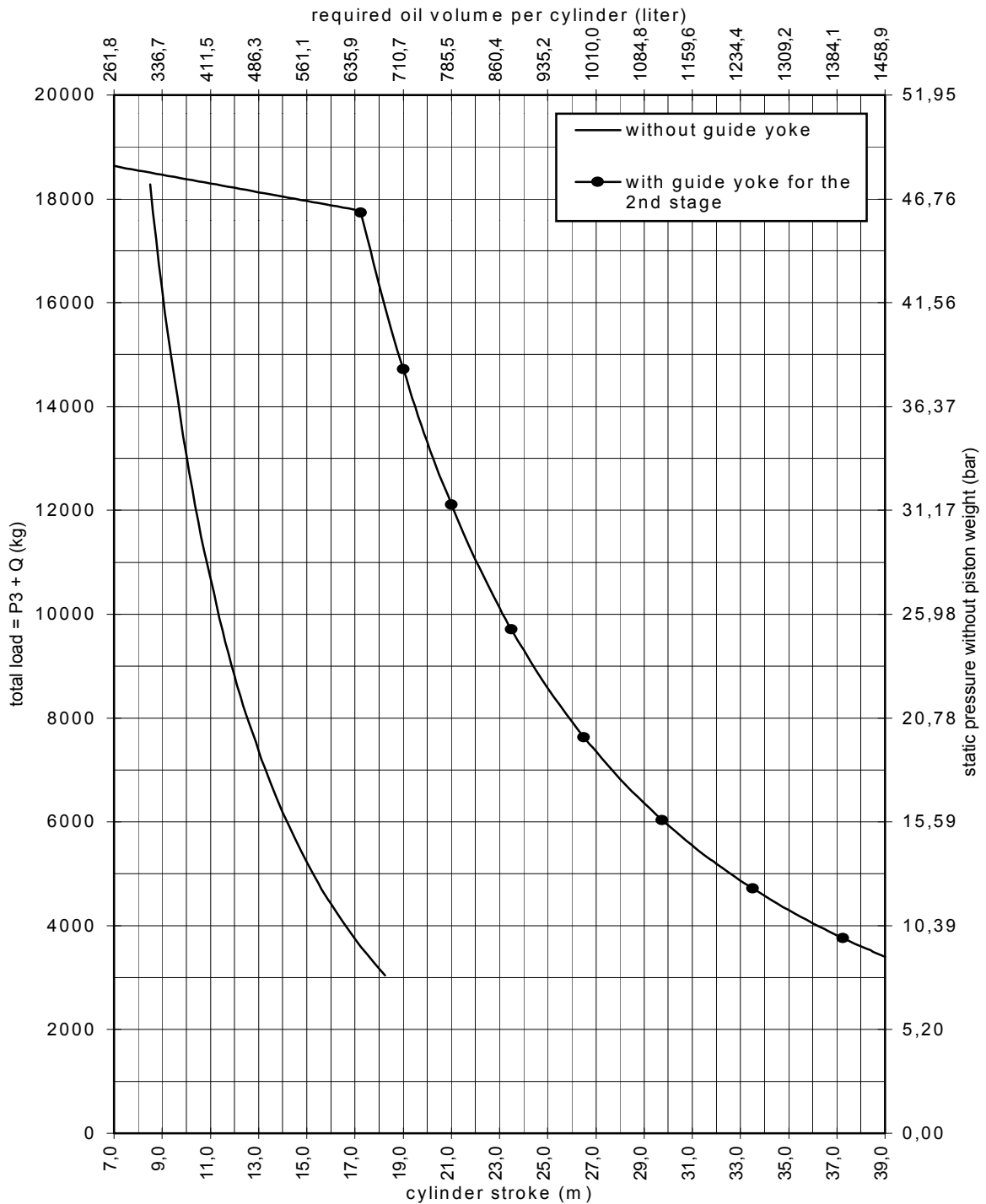


Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 120/3 - VE



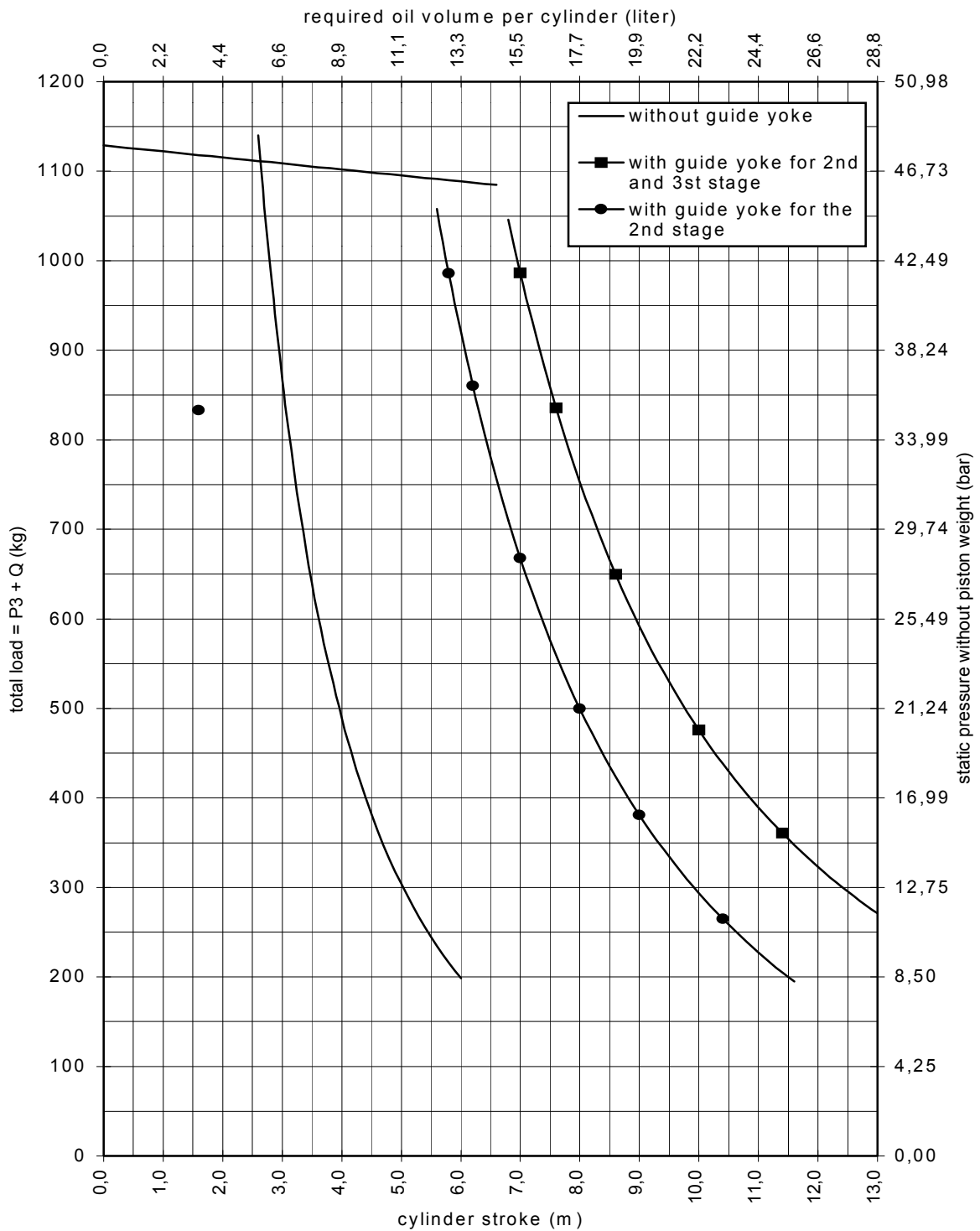
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 51 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 120 / 170 / 240 (mm)	Factor of excess pressure = 1,4
reference area	A = 264,365 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 298,530 (kg) (0 stroke) + 57,671 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Central Direct System  
Type 3PL 140/3 - VE



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	$p = 51$ (bar)	Factor of safety to buckling = 2,0
piston rod diameter	$d_a = 140 / 200 / 290$ (mm)	Factor of excess pressure = 1,4
reference area	$A = 377,655$ (cm <sup>2</sup> )	$Q =$ pay load (kg)
weight of piston	$m_p = 441,973$ (kg) (0 stroke) + 83,981 (kg) (per meter stroke)	$P_3 =$ weight of cabina (kg) $z =$ number of cylinders

Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 35/3 - RS

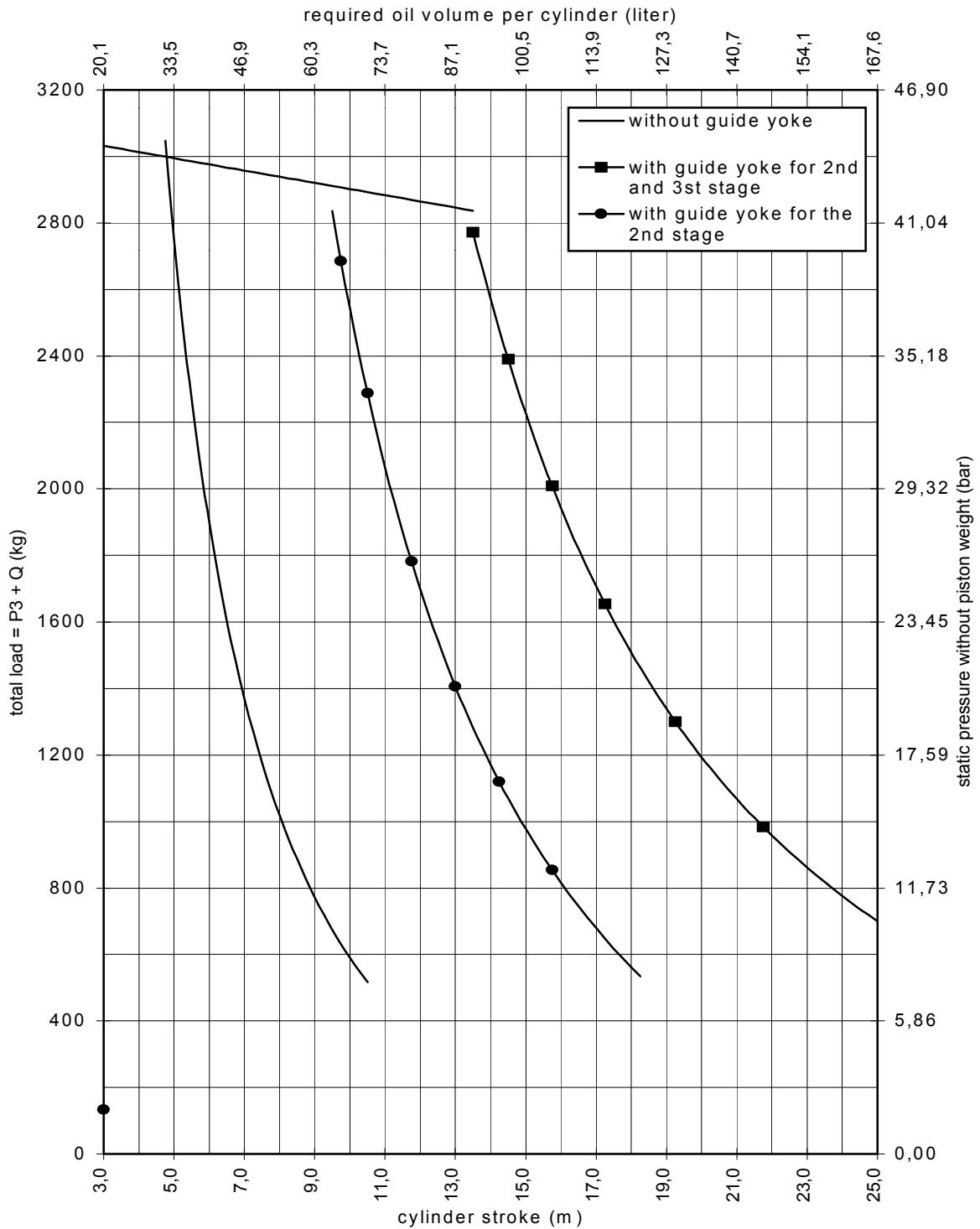


Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 49 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 35 / 50 / 70 (mm)	Factor of excess pressure = 1,4
reference area	A = 23,090 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 24,465 (kg) (0 stroke) + 6,702 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders



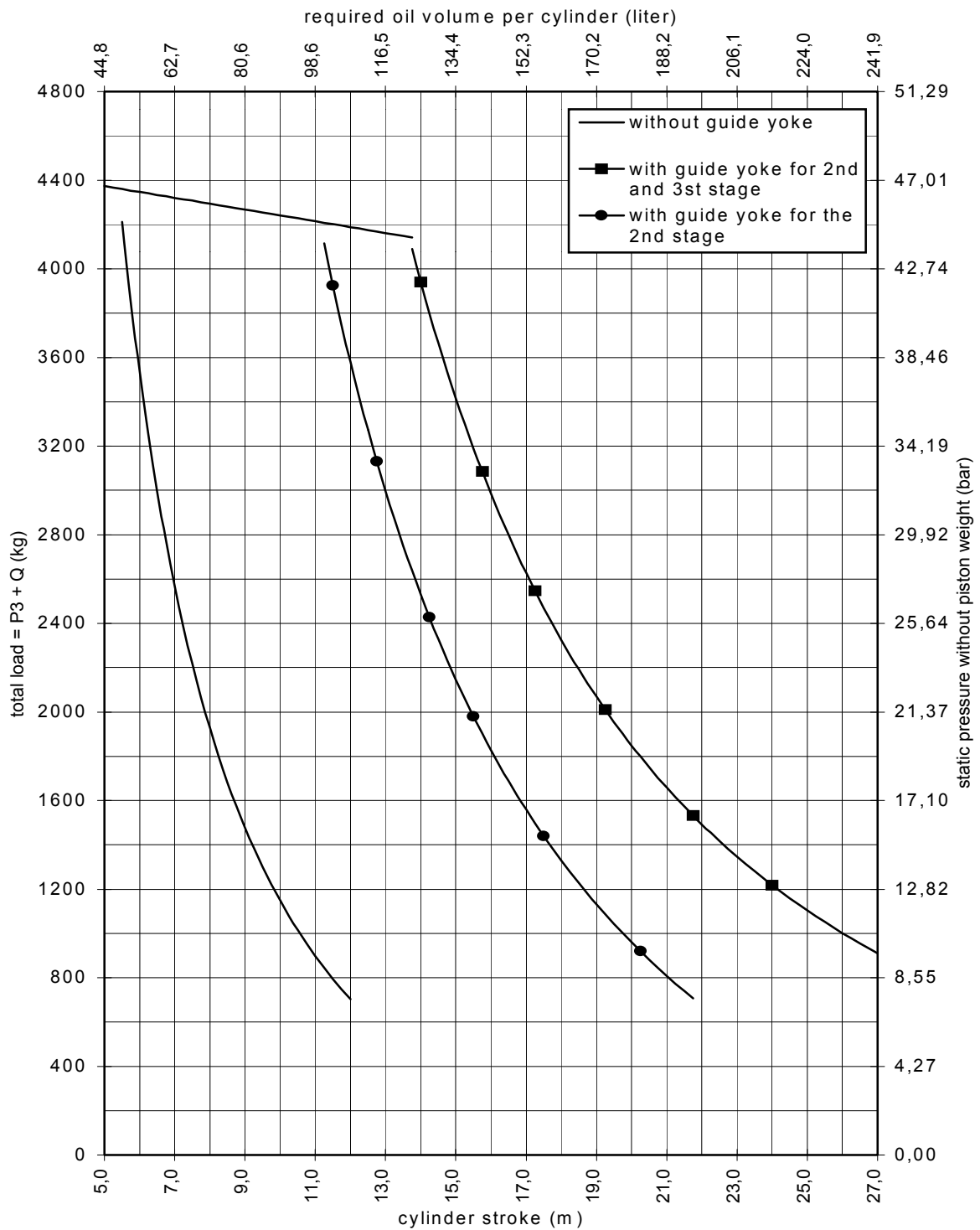


Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 63/3 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	$p = 46$ (bar)	Factor of safety to buckling = 2,0
piston rod diameter	$d_a = 63 / 85 / 120$ (mm)	Factor of excess pressure = 1,4
reference area	$A = 66,928$ (cm <sup>2</sup> )	$Q =$ pay load (kg)
weight of piston	$m_p = 50,866$ (kg) (0 stroke) + 18,556 (kg) (per meter stroke)	$P_3 =$ weight of cabina (kg)
		$z =$ number of cylinders

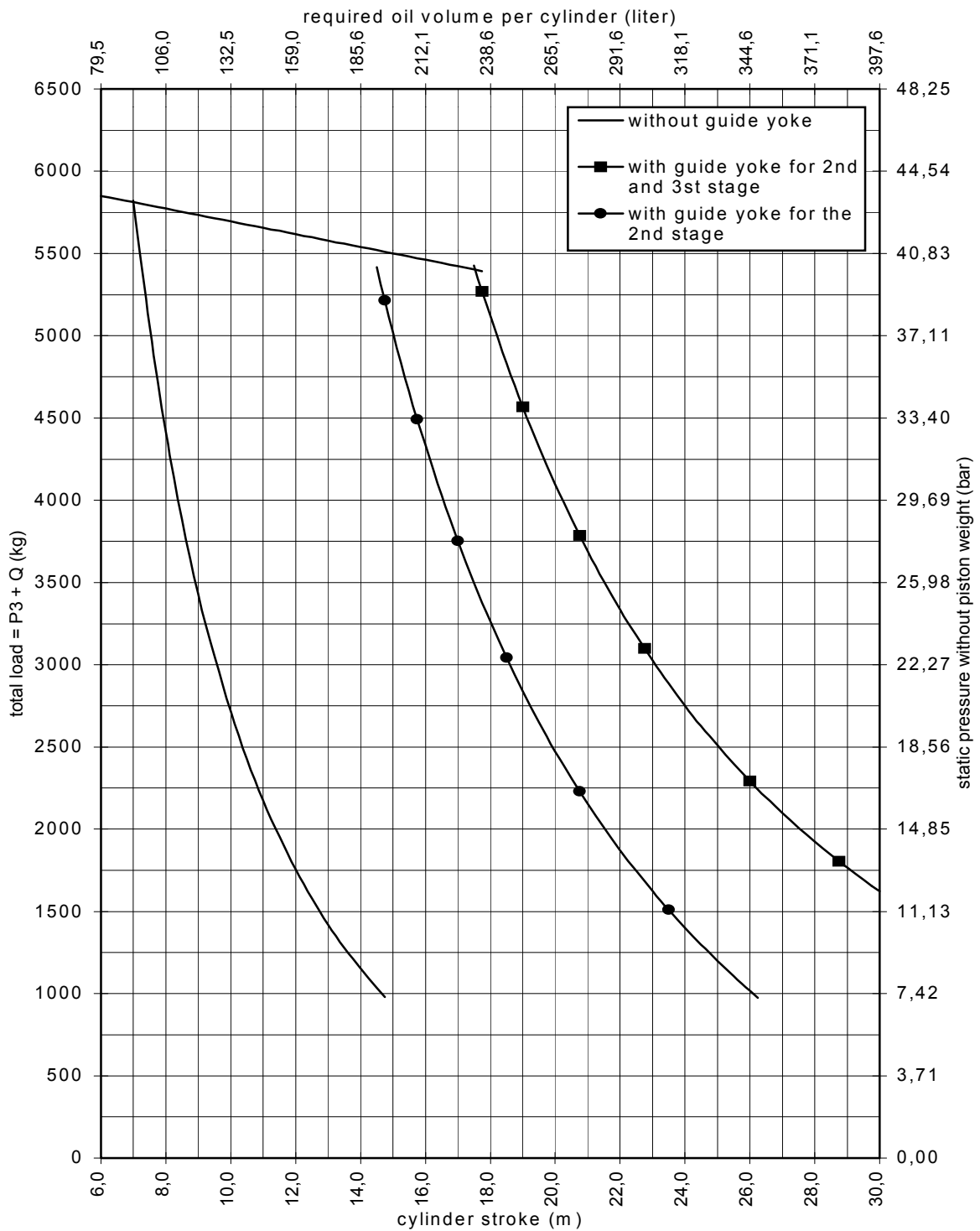
Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 70/3 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 49 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 70 / 100 / 140 (mm)	Factor of excess pressure = 1,4
reference area	A = 91,816 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 80,530 (kg) (0 stroke) + 26,460 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders



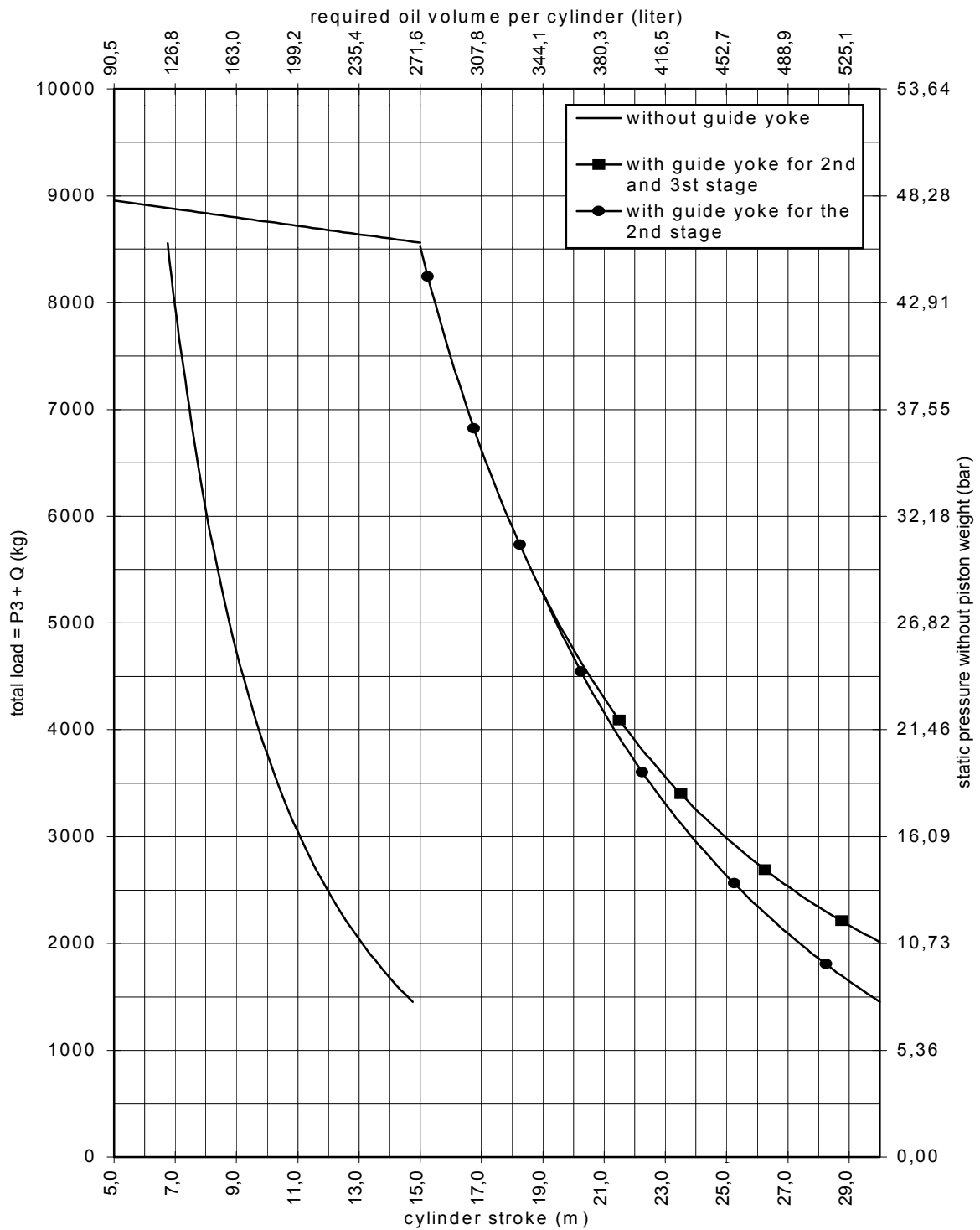
Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 85/3 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 46 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 85 / 100 / 170 (mm)	Factor of excess pressure = 1,4
reference area	A = 132,161 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 113,149 (kg) (0 stroke) + 38,875 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

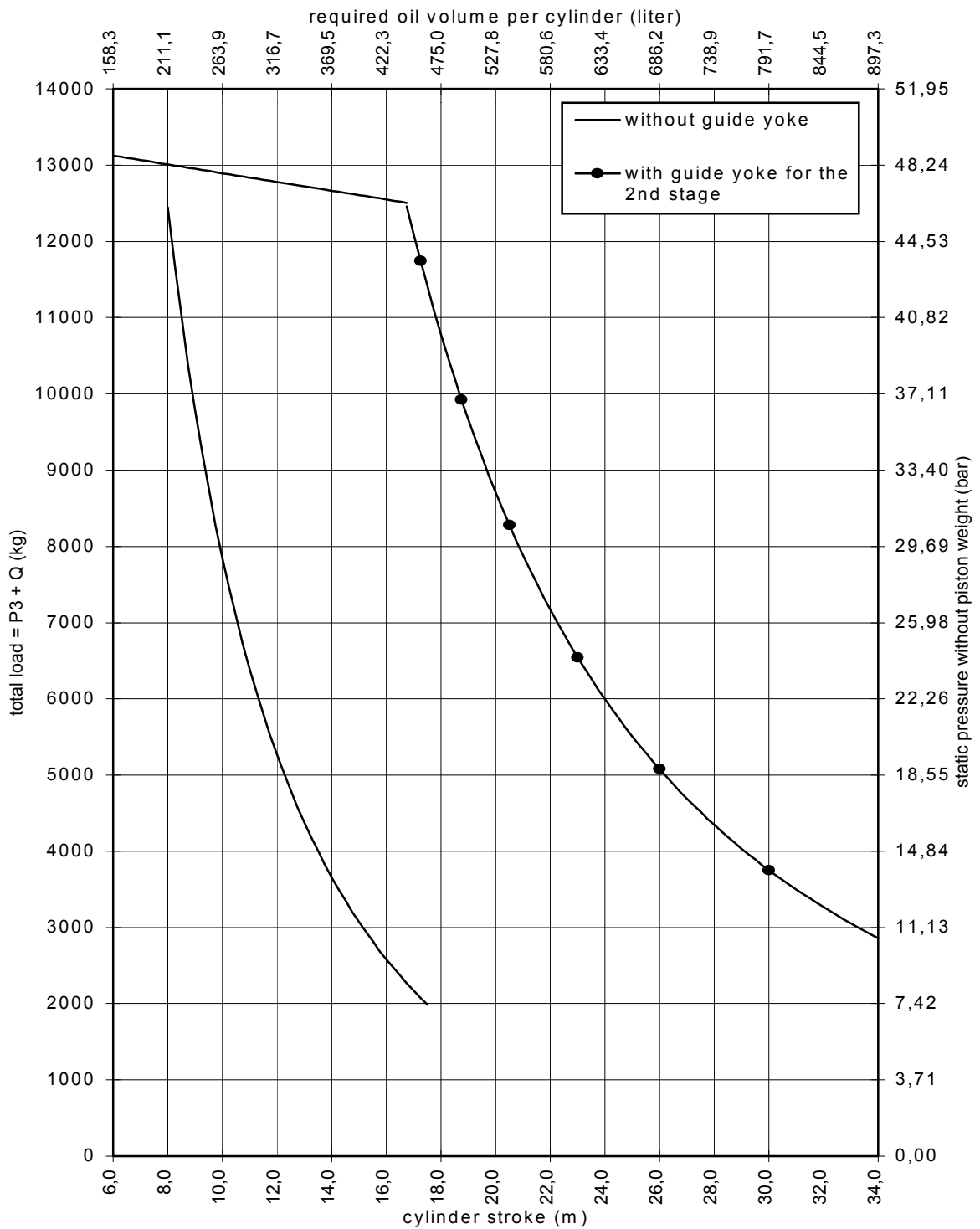
C.O.A.M. S.p.A. COMPONENTI OLEODINAMICI PER ASCENSORI E MONTACARICHI	Selection diagram for telescopic 3-stage cylinder - Side Ram System	emesso	S. A.	20/11/1998								
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Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 100/3 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 50 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 100 / 140 / 200 (mm)	Factor of excess pressure = 1,4
reference area	A = 182,891 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 168,195 (kg) (0 stroke) + 39,491 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 120/3 - RS



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 51 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 120 / 170 / 240 (mm)	Factor of excess pressure = 1,4
reference area	A = 264,365 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 273,191 (kg) (0 stroke) + 57,671 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

**C.O.A.M.** S.p.A.  
COMPONENTI OLEODINAMICI PER  
ASCENSORI E MONTACARICHI

Selection diagram for telescopic  
3-stage cylinder - Side Ram System

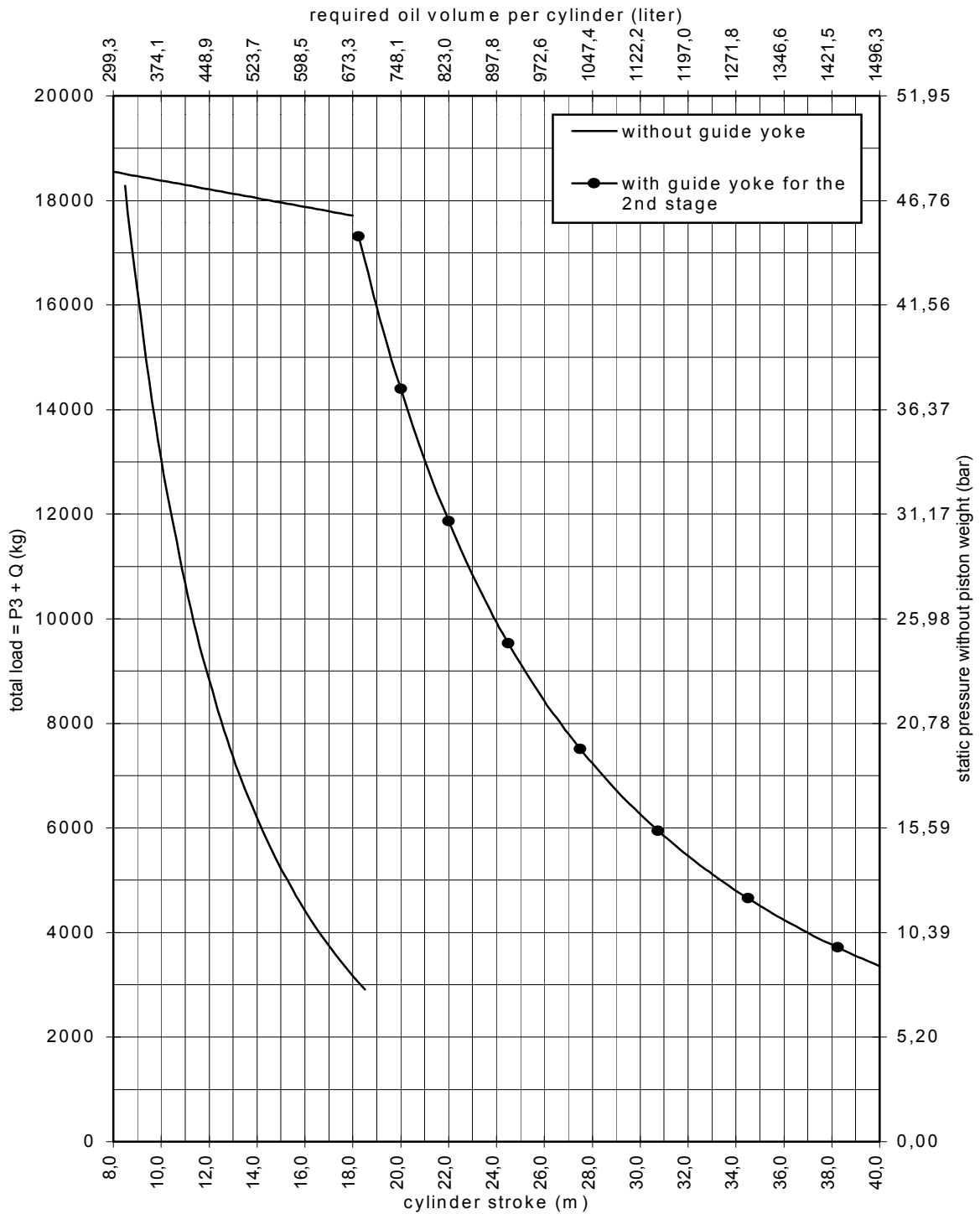
emesso	S. A.	20/11/1998
controllato		20/11/1998
nullaosta		

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DOCUMENTAZIONI TECNICHE

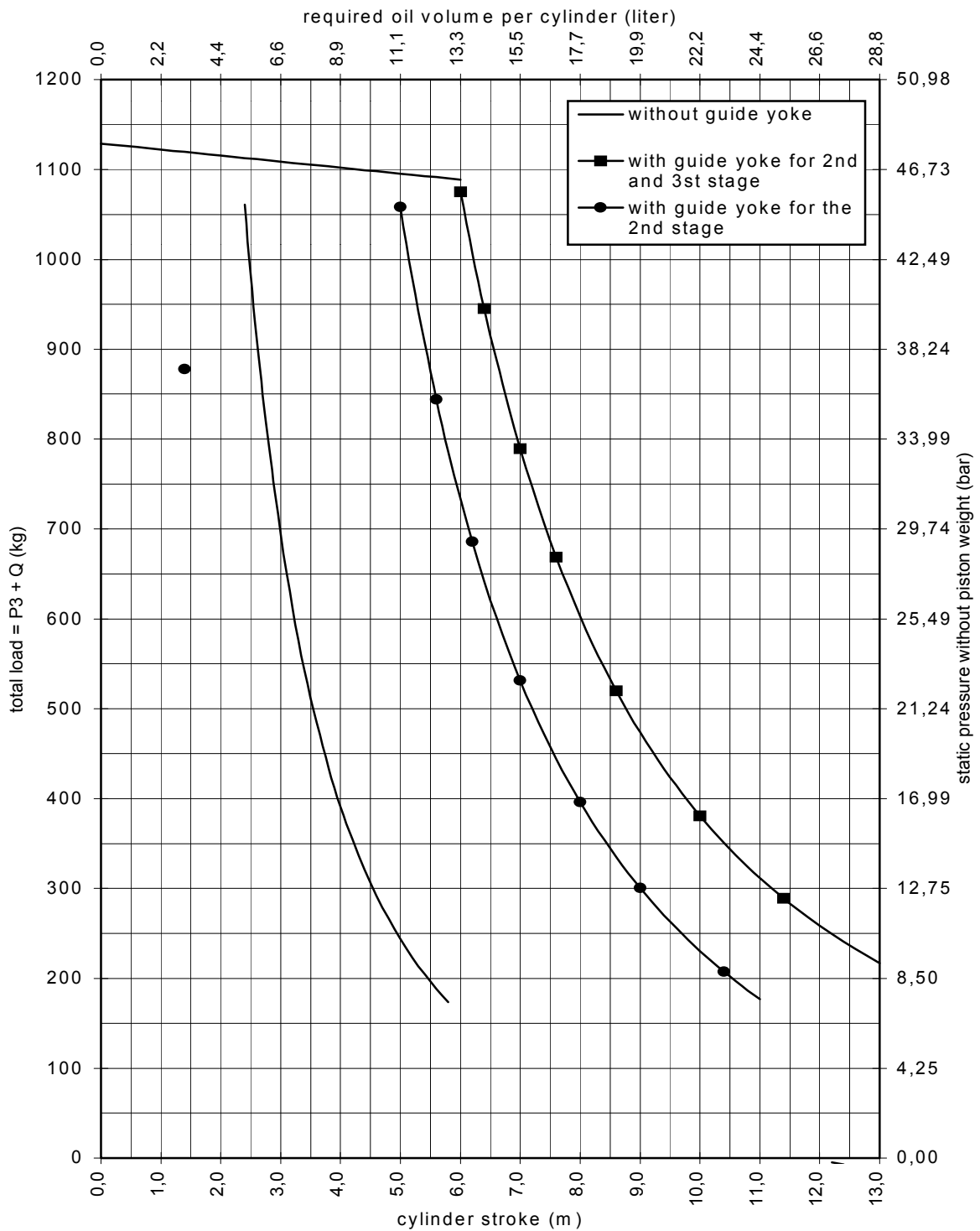
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Selection diagram for Telescopic 3-stage Cylinder  
Acc. to EN 81/2 - Side Ram System  
Type 3PL 140/3 - RS



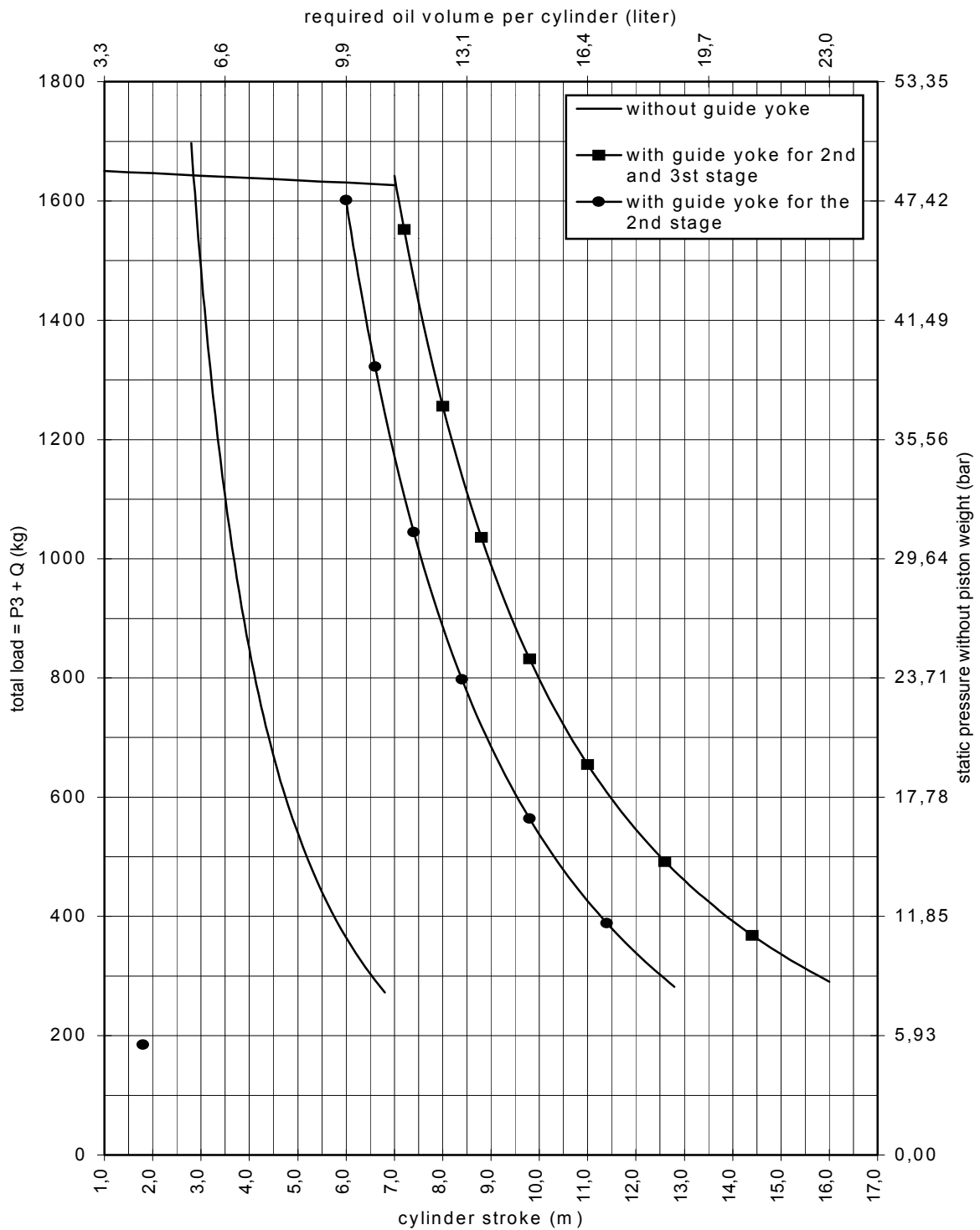
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 51 (bar)	Factor of safety to buckling = 2,0
piston rod diameter	d <sub>a</sub> = 140 / 200 / 290 (mm)	Factor of excess pressure = 1,4
reference area	A = 377,655 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 411,824 (kg) (0 stroke) + 83,981 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg) z = number of cylinders

Selection diagram for Telescopic 3-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 35/3 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 49 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 35 / 50 / 70 (mm)	Factor of excess pressure = 1,4
reference area	A = 23,090 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 24,465 (kg) (0 stroke) + 6,702 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg) z = number of cylinders

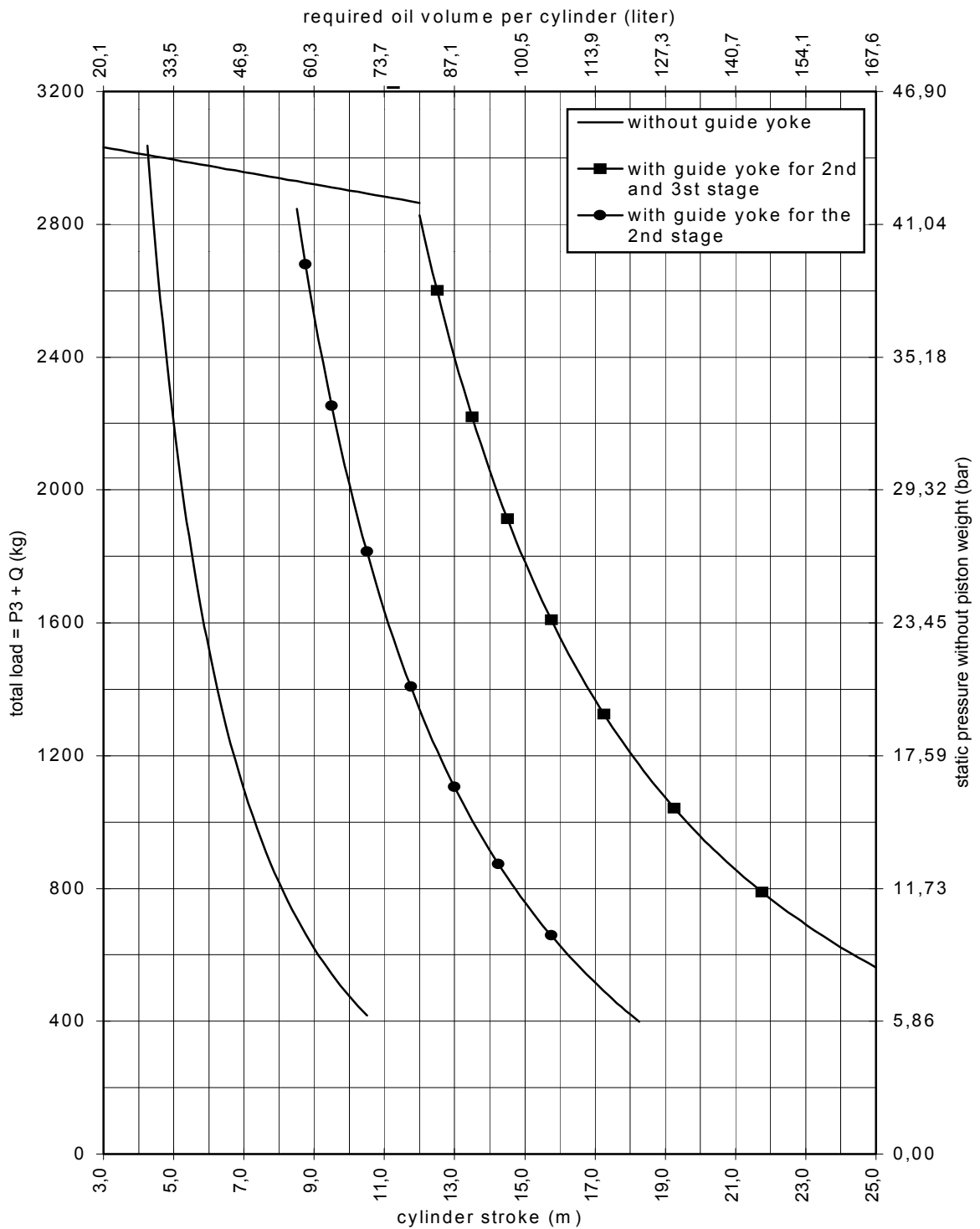
Selection diagram for Telescopic 3-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 42/3 - VT



Technical data :	$p_{stat} = 0,981 \cdot \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	$p = 50$ (bar)	Factor of safety to buckling = 2,5
piston rod diameter	$d_a = 42 / 60 / 85$ (mm)	Factor of excess pressure = 1,4
reference area	$A = 33,100$ (cm <sup>2</sup> )	$Q =$ pay load (kg)
weight of piston	$m_p = 27,014$ (kg) (0 stroke) + 9,782 (kg) (per meter stroke)	$P_3 =$ weight of cabina (kg)
		$z =$ number of cylinders



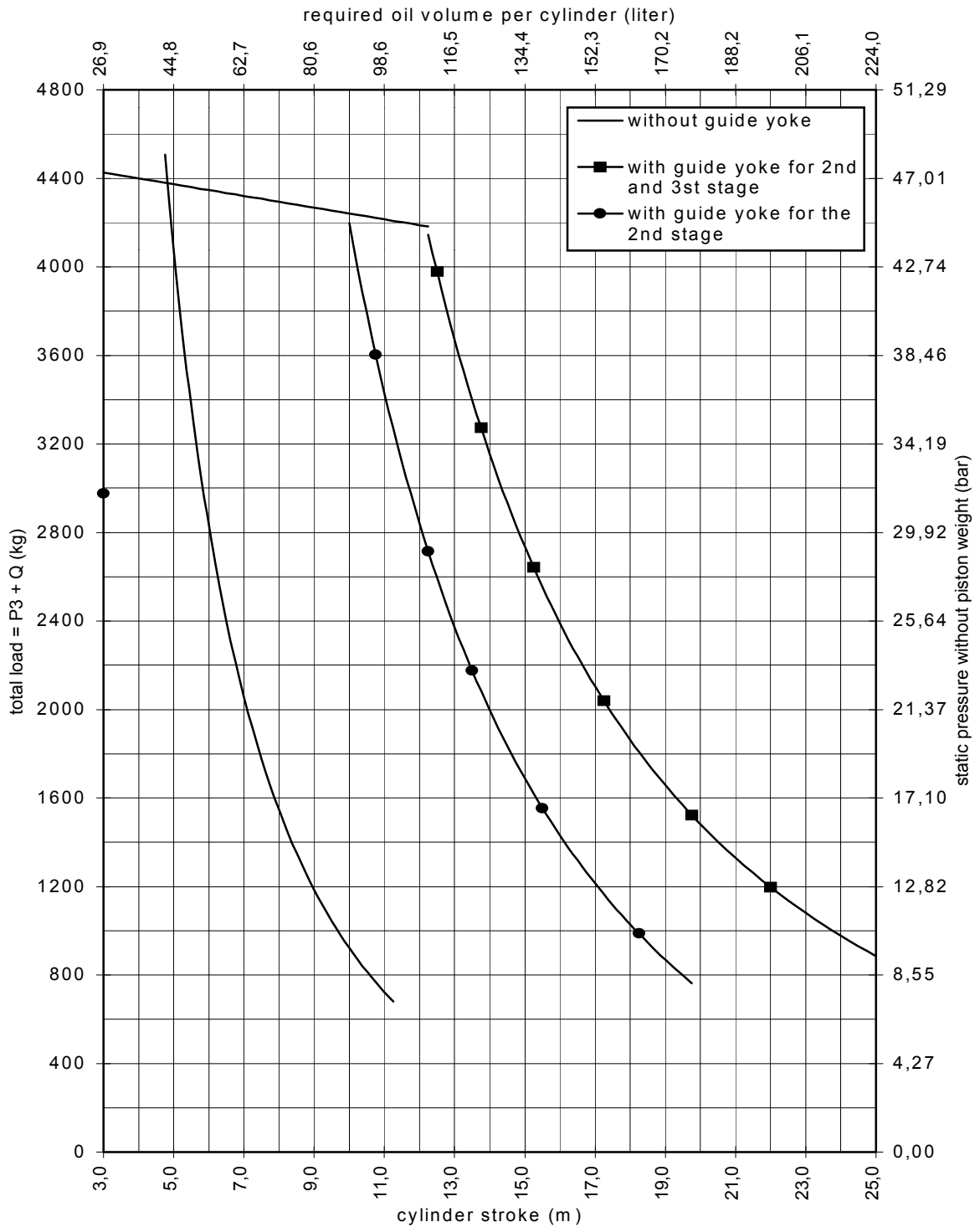
Selection diagram for Telescopic 3-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 63/3 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 46 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 63 / 85 / 120 (mm)	Factor of excess pressure = 1,4
reference area	A = 66,928 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 50,866 (kg) (0 stroke) + 18,556 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg) z = number of cylinders

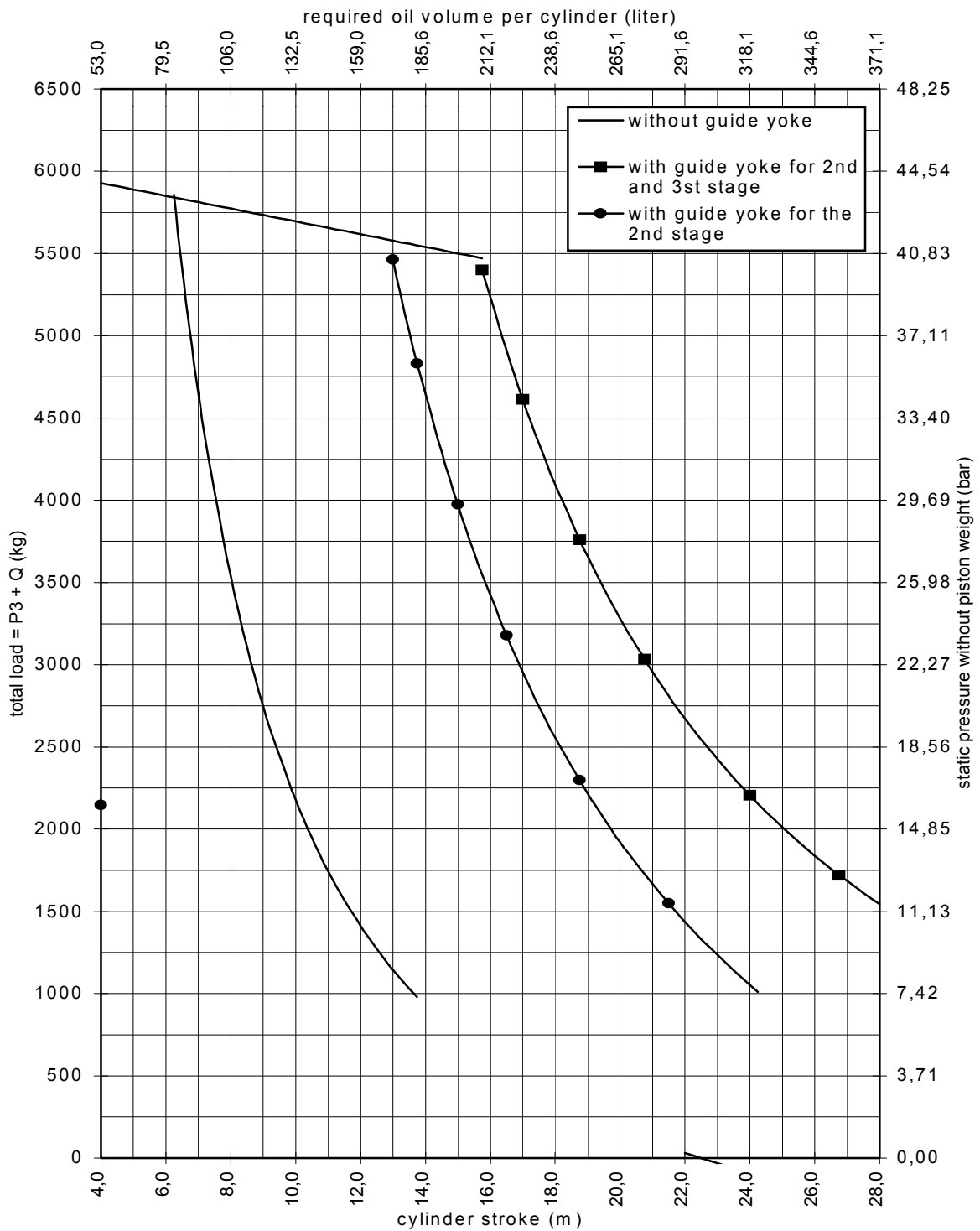


Selection diagram for Telescopic 3-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 70/3 - VT



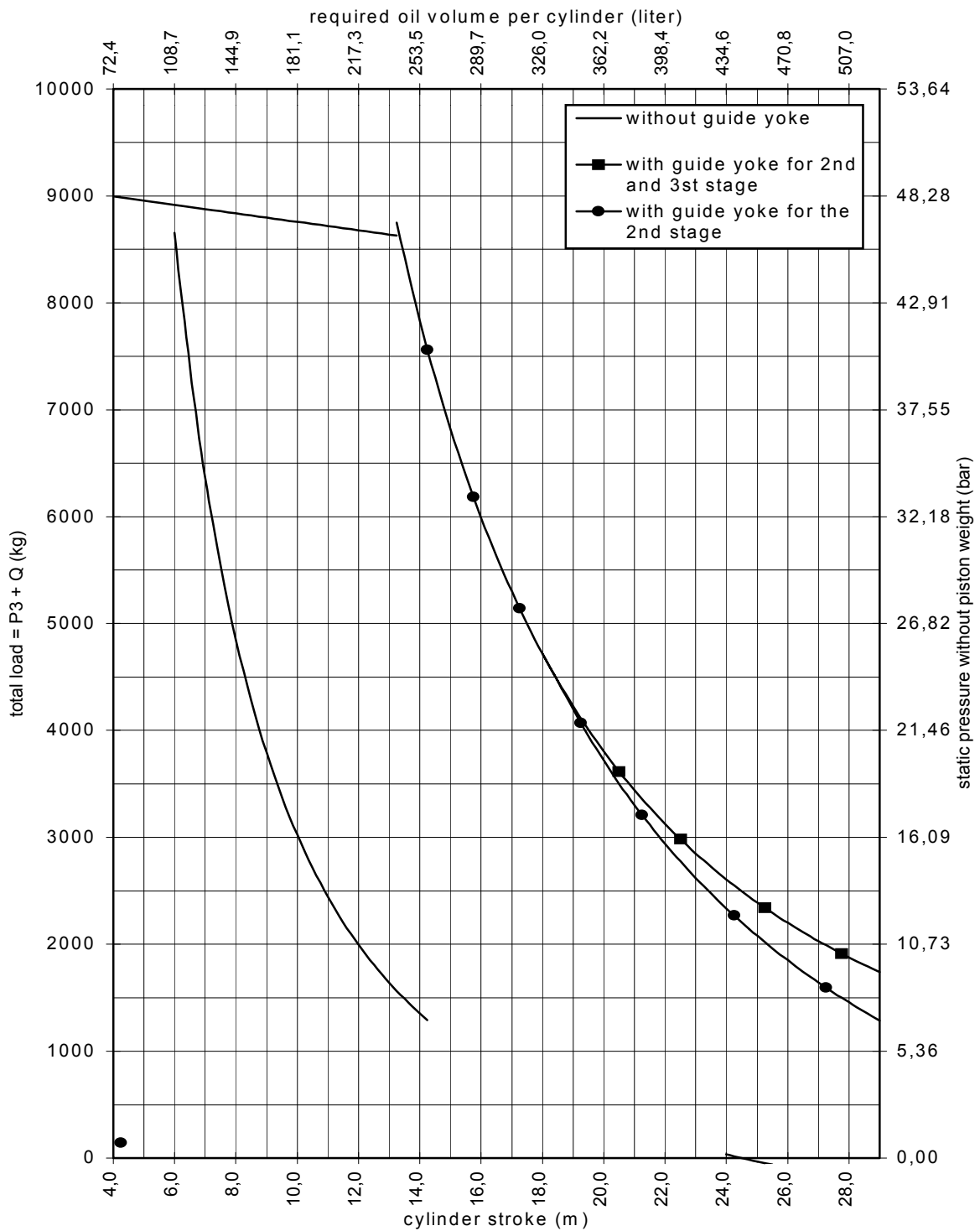
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 49 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 70 / 100 / 140 (mm)	Factor of excess pressure = 1,4
reference area	A = 91,816 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 80,530 (kg) (0 stroke) + 26,460 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg) z = number of cylinders

Selection diagram for Telescopic 3-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 85/3 - VT



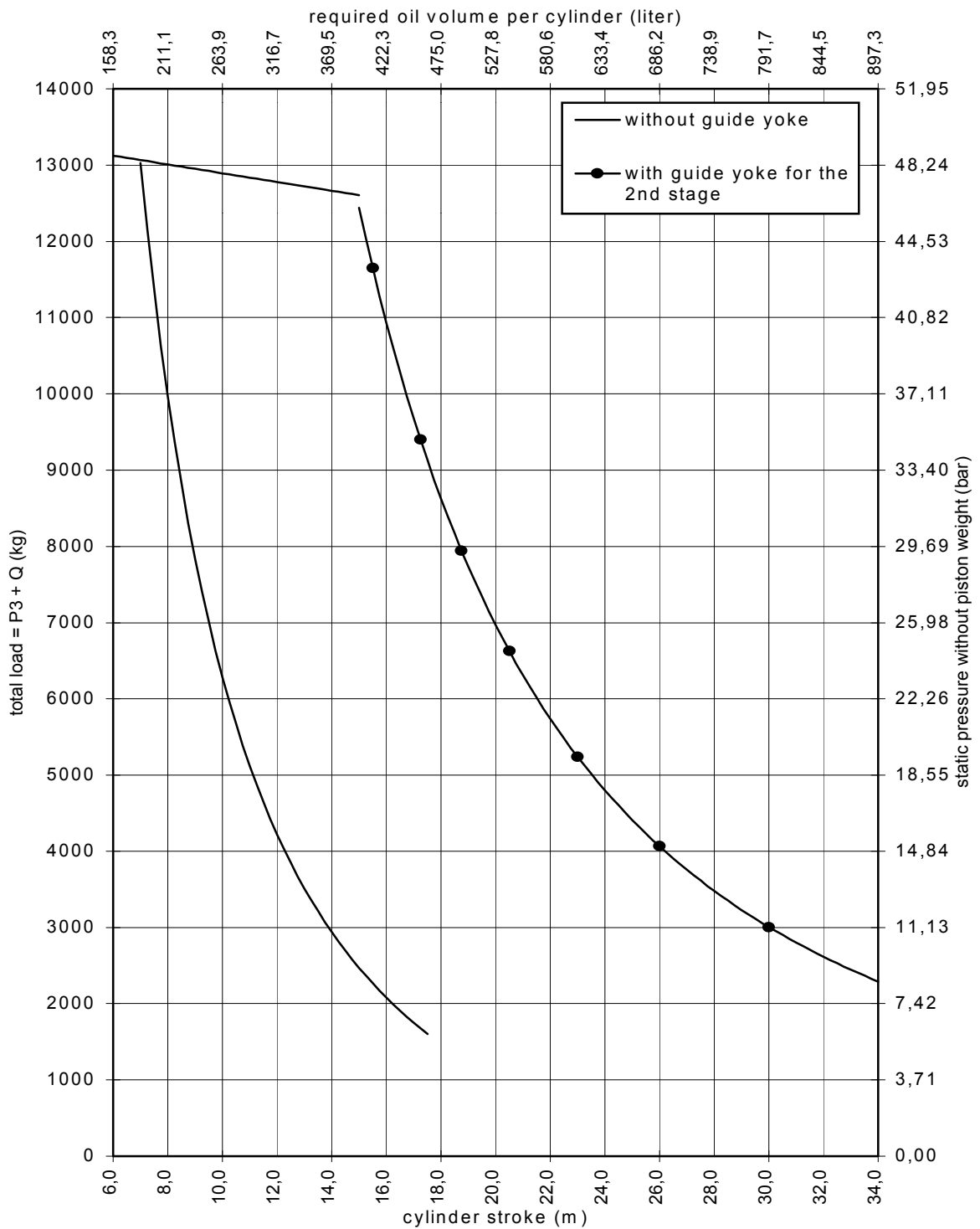
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 46 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 85 / 100 / 170 (mm)	Factor of excess pressure = 1,4
reference area	A = 132,161 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 113,149 (kg) (0 stroke) + 38,875 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg) z = number of cylinders

Selection diagram for Telescopic 3-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 100/3 - VT



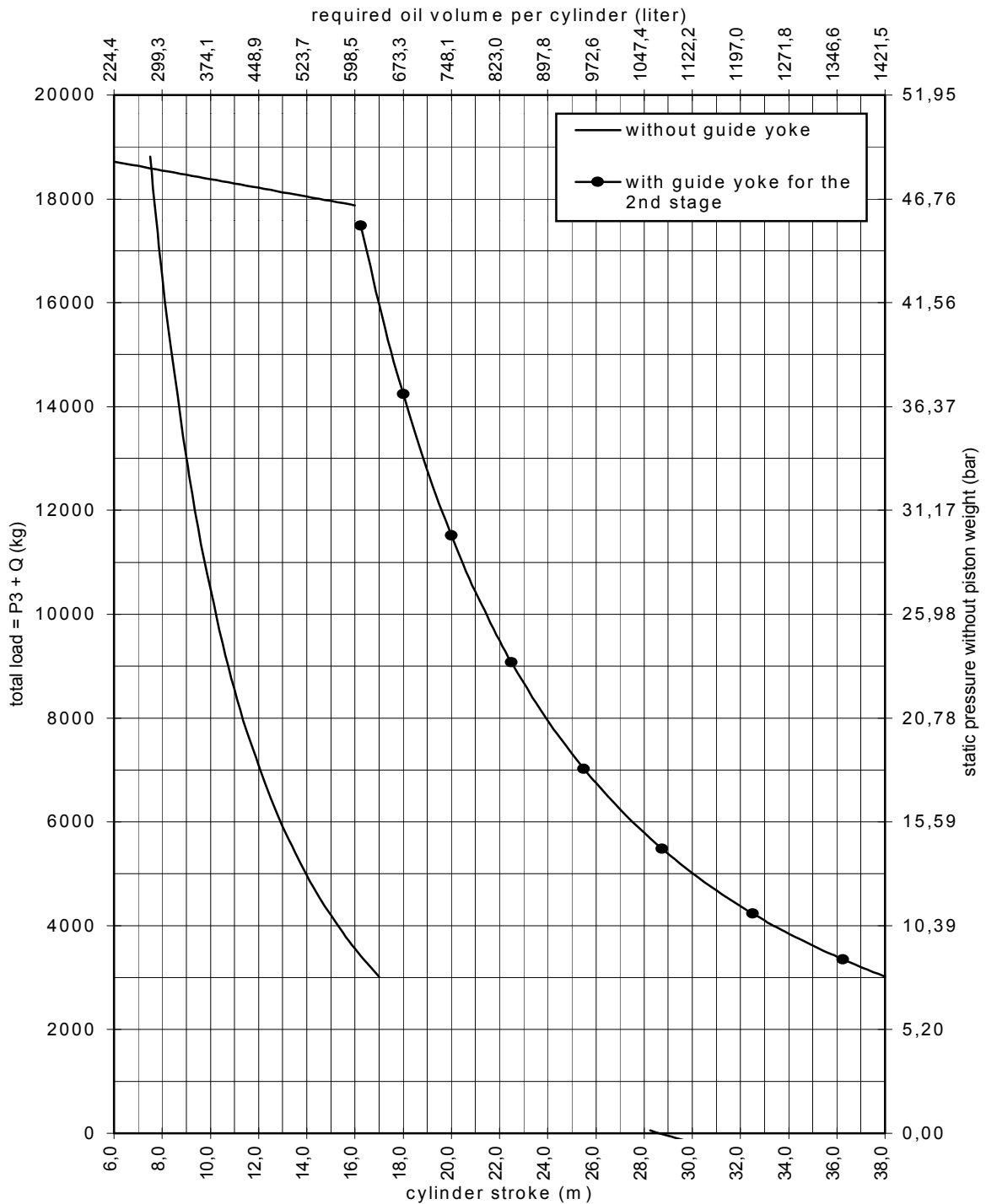
Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 50 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 100 / 140 / 200 (mm)	Factor of excess pressure = 1,4
reference area	A = 182,891 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 168,195 (kg) (0 stroke) + 39,491 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 3-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 120/3 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	p = 51 (bar)	Factor of safety to buckling = 2,5
piston rod diameter	d <sub>a</sub> = 120 / 170 / 240 (mm)	Factor of excess pressure = 1,4
reference area	A = 264,365 (cm <sup>2</sup> )	Q = pay load (kg)
weight of piston	m <sub>p</sub> = 273,191 (kg) (0 stroke) + 57,671 (kg) (per meter stroke)	P <sub>3</sub> = weight of cabina (kg)
		z = number of cylinders

Selection diagram for Telescopic 3-stage Cylinder  
Acc. to TRA 200 - Central Direct and Side Ram System  
Type 3PL 140/3 - VT



Technical data :	$p_{stat} = 0,981 * \frac{P_3 + Q + m_p}{A + z}$ (bar)	
max static pressure	$p = 51$ (bar)	Factor of safety to buckling = 2,5
piston rod diameter	$d_a = 140 / 200 / 290$ (mm)	Factor of excess pressure = 1,4
reference area	$A = 377,655$ (cm <sup>2</sup> )	$Q =$ pay load (kg)
weight of piston	$m_p = 411,824$ (kg) (0 stroke) + 83,981 (kg) (per meter stroke)	$P_3 =$ weight of cabina (kg) $z =$ number of cylinders